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CS24-43 Screw Threads and Tap-Drill Sizes

IAR 22 1943 U. S. DEPARTMENT OF COMMERCE

JESSE H. JONES, Secretary

NATIONAL BUREAU OF STANDARDS

LYMAN J. BRIGGS, Director

SCREW THREADS AND TAP-DRILL SIZES

COMMERCIAL STANDARD CS24-43

(Revision and consolidation of CS24-30 and CS25-30)

Effective Date for New Production from February 10, 1943



A RECORDED VOLUNTARY STANDARD OF THE TRADE

UNITED STATES
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PROMULGATION

of

COMMERCIAL STANDARD CS24-43

for

SCREW THREADS AND TAP-DRILL SIZES

(Revision and consolidation of CS24-30 and CS25-30)

At the request of the National Screw Thread Commission, American National screw-thread tables for shop use were circulated January 23, 1930, as recommended commercial standards to producers, distributors, and users for a written acceptance. They were subsequently accepted in writing by the industry and published under the titles, American National Standard Screw Threads, Coarse and Fine-Thread Series, Commercial Standard CS24-30; and American National Special Screw Threads, Commercial Standard CS25-30.

On July 28, 1942, on the recommendation of the Interdepartmental Screw Thread Committee, and with the endorsement of the standing committee, a consolidation and revision of CS24-30 and CS25-30, under the title of Recommended Commercial Standard for Screw Threads and Tap-Drill Sizes, was circulated for acceptance. Those concerned have since accepted and approved the standard as shown herein for promulgation by the United States Department of Commerce, through the National Bureau of Standards.

The standard is effective for new production from February 10, 1943.

Promulgation recommended.

I. J. Fairchild, Chief, Division of Trade Standards.

Promulgated.

Lyman J. Briggs, Director, National Bureau of Standards.

Promulgation approved.

Jesse H. Jones, Secretary of Commerce.

ERRATA SHEET SCREW THREADS AND TAP-DRILL SIZES

COMMERCIAL STANDARD 0824-43

PAGE	LINE	NOW READS	SHOULD READ
-51	8 OF LAST COLUMN	.3086	.4086
23	B OF COLUMN 3	5.305	.5305
26	20 OF COLUMN I	1 1/8	1 7/8
26	4 FROM BOTTOM, COL. 1	54/14	5 1/4
27	2 OF COLUMN I	3/16	13/16
27	4 OF COLUMN 1	5/16	15/16
27	14 OF COLUMN 5	1,4183	1.4813
31 -	7 OF COLUMN 3	0.0006	0.0056
31 31	II OF COLUMN I	'1N.6	MIN.5
35 35 35	HEADING, COLUMN 3	17/16	1 7/16
35	HEADING, COLUMN 9	13/16	1 1/3/16
35	13 OF CÓLUMM 10	1.8150	1.8153
45	2 FROM BOTTOM, COL. I	5/16 F	5/16
46	3 FROM BOTTOM, COL. 6	9/32 IN.	1'9/32 IN.
46 46	2 FROM BOTTOM, COL. 6	19/64 IN.	1 19/64 14.
46	3 FROM BOTTOM, COL. 8	8	87
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SCREW THREADS AND TAP-DRILL SIZES

(Revision and Consolidation of CS24-30 and CS25-30)

COMMERCIAL STANDARD CS24-43

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PURPOSE

1. The purpose of this standard is to make available for convenient shop use and acceptance inspection the essential specifications, definitions, and dimensional data on screw threads and tap drills, which are recorded more completely in "Screw Thread Standards for Federal Services, 1942," National Bureau of Standards Handbook H28.

SCOPE

2. This standard covers the predominating sizes of American National Screw Threads in the following series and fits, with the corresponding tap-drill sizes:

Coarse-thread series, sizes No. 1 (0.073") to 4", classes 1, 2, 3, and

Fine-thread series, sizes No. 0 (0.060") to $1\frac{1}{2}$ ", classes 1, 2, 3, and 4 fits.

8-pitch-thread series, sizes 1" to 6", classes 2 and 3 fits. 12-pitch-thread series, sizes ½" to 6", classes 2 and 3 fits. 16-pitch-thread series, sizes ¾" to 4", classes 2 and 3 fits. To-pitch-thread series, sizes ¾" to 4", classes 2 and 3 fits. Extra-fine-thread series, sizes ¼" to 2", classes 2 and 3 fits.

Tap drills for No. 1 to 3¾" coarse-thread series. Tap drills for No. 0 to 1½" fine-thread series. Tap drills for 1" to 3½" 8-pitch-thread series.

Tap drills for ½" to 3½" 12-pitch-thread series.

Tap drills for ¾" to 3½" 16-pitch-thread series. Tap drills for ¼" to 2" extra-fine-thread series.

DEFINITIONS

3. Terms relating to screw threads and illustrations of terminology.

3a. Numbering of tables and figures.—Since most of the figures and tables herein are identical with those in National Bureau of Standards Handbook H28, they are numbered identically for convenient cross reference, even though this results in some numerical discontinuity in this standard. Figures 1, 2, 3, and 10 illustrate the terms and symbols as defined.

3b. Screw thread.—A ridge of uniform section in the form of a helix on the external or internal surface of a cylinder, or in the form of a

conical spiral on the external or internal surface of a cone.

3c. External and internal threads. 1—An external thread is a thread on the outside of a member. Example: A threaded plug.

An internal thread is a thread on the inside of a member. Example:

A threaded hole.

3d. Major diameter.—The largest diameter of the thread of the screw or nut. The term "major diameter" replaces the term "outside diameter" as applied to the thread of a screw and also the term "full diameter" as applied to the thread of a nut.

3e. Minor diameter.—The smallest diameter of the thread of the screw or nut. The term "minor diameter" replaces the term "core diameter" as applied to the thread of a screw and also the term

"inside diameter" as applied to the thread of a nut.

3f. Pitch diameter. On a straight screw thread, the diameter of an imaginary cylinder, the surface of which would pass through the threads at such points as to make equal the width of the threads and the width of the spaces cut by the surface of the cylinder. On a taper screw thread, the diameter, at a given distance from a reference plane perpendicular to the axis of an imaginary cone, the surface of which would pass through the threads at such points as to make equal the width of the threads and the width of the spaces cut by the surface of the cone.

3g. Pitch.—The distance from a point on a screw thread to a corresponding point on the next thread measured parallel to the

axis.

The pitch, in inches,= Number of threads per inch

3h. Lead.—The distance a screw thread advances axially in one turn. On a single-thread screw the lead and pitch are identical; on

These terms are here defined because of possible confusion arising from the fact that an "internal member" has an "external thread," and vice versa. For the sake of brevity, an external thread is hereinafter referred to as a "screw," and an internal thread as a "nut."

a double-thread screw the lead is twice the pitch; on a triple-thread screw the lead is three times the pitch, etc.

3i. Angle of thread.—The angle included between the sides of the

thread measured in an axial plane.

3j. Half angle of thread.—The angle included between a side of the thread and the normal to the axis, measured in an axial plane.

3k. Helix angle.—The angle made by the helix, or conical spiral, of the thread at the pitch diameter with a plane perpendicular to the axis.

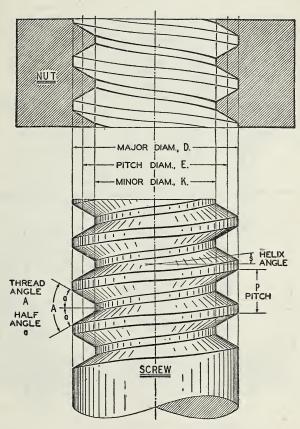


FIGURE 1.—Screw-thread notation.

3l. Crest.—The surface of the thread corresponding to the major diameter of the screw and the minor diameter of the nut.

3m. Root.—The surface of the thread corresponding to the minor diameter of the screw and the major diameter of the nut.

3n. Side or flank.—The surface of the thread which connects the crest with the root.

30. Axis of a screw.—The longitudinal central line through the screw.

3p. Base of thread.—The bottom section of the thread; the greatest section between the two adjacent roots.

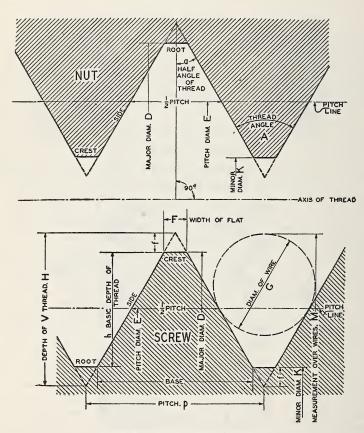
3q. Depth of thread.—The distance between the crest and the base of the thread measured normal to the axis.

3r. Number of threads.—Number of threads in 1 inch of length.

3s. Length of engagement.—The length of contact between two mated parts, measured axially.
3t. Depth of engagement.—The depth of thread contact of two

mated parts, measured radially.

3u. Pitch line.—An element of the imaginary cylinder or cone specified in definition 3f.



FIGURE, 2-Screw-thread notation.

3v. Thickness of thread.—The distance between the adjacent sides

of the thread measured along or parallel to the pitch line.

3w. Mean area.—The term "mean area of a screw", when used in specifications and for other purposes, designates the cross-sectional area computed from the mean of the basic pitch and minor diameters.

4. Terms relating to classes of fit and tolerances.

4a. Allowance.—An intentional difference in the dimensions of mating parts. It is the minimum clearance or the maximum interference which is intended between mating parts. It represents the

condition of the tightest permissible fit, or the largest internal member mated with the smallest external member. Example:

One-half inch, class 1 fit, American National coarse-thread series: Minimum pitch diameter of nut Maximum pitch diameter of screw	
Allowance (positive)One-half inch, class 4 fit, American National coarse thread series:	0. 0022
Minimum pitch diameter of nut	0. 4500
Maximum pitch diameter of screw	
Allowance (negative)	0, 0004

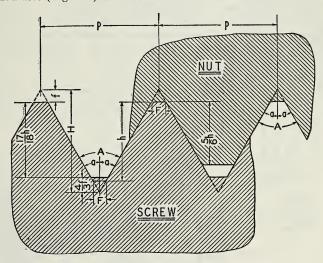


FIGURE 3.—American National form of thread.

Note.—No allowance is shown. This condition exists in classes 2 and 3 fits, where both the minimum nut and the maximum screw are basic.

```
\begin{array}{c} \text{NOTATION} \\ a=30^{\circ} \\ a=30^{\circ} \\ n=\text{number of threads per inch} \\ H=0.866025 \ p=\text{depth of 60^{\circ} sharp V thread} \\ h=0.649519 \ p=\text{depth of American National form of thread} \\ 56h=0.541266 \ p=\text{maximum depth of engagement} \\ 17/18h=0.613435 \ p\\ F=0.125000 \ p=\text{width of flat at crest and root of American National form} \\ f=0.10823 \ p\\ =\frac{1}{4}H \\ =\frac{1}{4}(h) \end{array} \right] = \text{depth of truncation}
```

4b. Tolerance.—The amount of variation permitted in the size of a part. Example:

One-half-inch screw, class 1 fit, American National coarse-thread series: Maximum pitch diameter	0.	4478 4404
Tolerance	0.	0074

4c. Basic size.—The theoretical, or nominal, standard size from which all variations are made.

4d. Crest clearance.—Defined on a screw form as the space between the crest of a thread and the root of its mating thread.

4e. Finish.—The character of the surface on a screw thread or other

product.

4f. Fit.—The relation between two mating parts with reference to the conditions of assembly, for example, classes 1, 2, 3, and 4. Each fit has its proper place, and none should be regarded as superior or

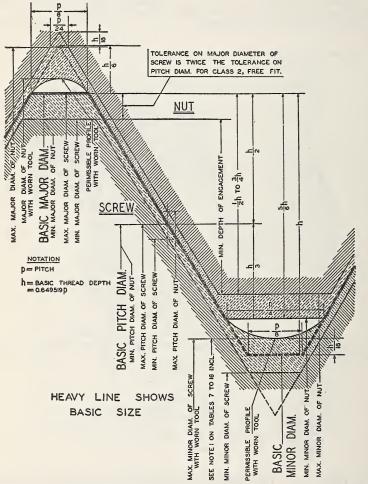


FIGURE 10.—Illustration of tolerances and crest clearances for class 3 fit.

inferior, provided that there is compliance with specification requirements under which it is manufactured and sold.

Class 1 fit includes screw-thread work in which the threads must assemble easily, and where an allowance is required to permit ready assembly, even when the threads are slightly bruised or dirty.

Class 2 fit represents a high quality of commercial screw-thread product and is recommended for the major portion of interchangeable screw-thread work, finished and semifinished bolts and nuts, machine screws, etc., where no allowance is required.

Class 3 fit is the same in every particular as class 2 fit except that the tolerances are smaller. The class 3 fit is intended to apply to interchangeable screw-thread work requiring the smallest practicable tolerances. Tapped holes within class 3 tolerances are difficult and

expensive to produce commercially.

Class 4 fit is designed for screw-thread work where extremely close tolerances are required. In the manufacture of screw-thread products to this class of fit, it will be necessary to use precision tools, gages made to special tolerances, and other refinements. This class of fit should, therefore, be used only in cases where the requirements of the mechanism being produced are exacting, or where there are special conditions which make this class of fit necessary. In order to ensure assembly with the degree of tightness desired, it may be necessary, in some cases, to select the parts when the product is being assembled.

4g. Neutral zone.—A positive allowance. (See Allowance, par. 4a.)
4h. Limits.—The extreme permissible dimensions of a part.

Example:

SYMBOLS

5. Symbols for designating screw-thread standards and thread dimensions are a necessity in commercial and engineering practice. The standardization of such symbols yields the usual advantages of standardization. Those listed below have been in customary use for many years, and their general use in standards, specifications, and textbooks is recommended.

6. Identification symbols.—These are for use on correspondence, drawings, shop and storeroom cards, specifications for parts, taps,

dies, gages, etc., and on tools and gages.

6a. The method of designating a screw thread by means of symbols is by the use of the initial letters of the thread series preceded by the diameter in inches (or the screw number) and number of threads per inch, all in Arabic characters, and followed by the classification of fit in Arabic numerals. If the thread is left hand, the symbol "LH" shall follow the class of fit. No symbol is used to distinguish right-hand threads. The number of threads per inch shall be indicated in all cases, irrespective of whether it is the standard number of threads for that particular size of threaded part or special. For screw threads of American National form but of special diameters, pitches, and lengths of engagement, the symbol "NS" shall be used. Examples:

American National coarse-thread series:	Mark
To specify a threaded part 1 inch in diameter, 8 threads	
per inch, class 1 fit	1''-8NC-1
Threaded part 1 inch in diameter, 8 threads per inch,	
class 2 fit, left hand	1''—8NC—2LH
American National fine-thread series:	
Threaded part 1 inch in diameter, 14 threads per inch,	
class 4 fit	1''—14NF—4
Threaded part \% inch in diameter, 18 threads per inch,	
class 5 fit	5/8''-18NF-5
Threaded part, 1/8 inch in diameter, 44 threads per inch,	, ,
class 2 fit	5-44NF-2
500932°432	
000002	

American National 8-, 12-, or 16-pitch-thread series:	Mark
Threaded part 1 inch in diameter, 12 threads per inch, class 3 fit	1''—12N—3
Threaded part 1½ inches in diameter, 8 threads per inch, class 2 fit, left hand	1½''—8N—2LH
American National extra-fine-thread series: Threaded part 1 inch in diameter, 20 threads per inch,	
class 3 fit	1''—20NEF—3
American National form, special pitch: Threaded part 1 inch in diameter, 18 threads per inch,	
class 2 fit	1''—18NS—2
Threaded part 1¼ inches in diameter, 20 threads per inch class 3 fit left hand	11/4"-20NS-3LH

SPECIFICATIONS

7. American National form of thread.—The form of thread profile specified herein, known previously as the "United States Standard or Sellers' profile", is adopted and shall hereafter be known as the "American National form of thread".

7a. Angle of thread.—The basic angle of thread (A, fig. 3) between the sides of the thread measured in an axial plane is 60°. The line bisecting this 60° angle is perpendicular to the axis of the screw thread.

7b. Flat at crest and root.—The flat at the root and crest of the basic

thread form is $\frac{1}{8} \times p$, or $0.125 \times p$.

7c. Depth of thread.—The depth of the basic thread form is

$$h = 0.649519 \times p$$
, or $h = \frac{0.649519}{n}$,

where

p = pitch in inches.

n= number of threads per inch.

h =basic depth of thread.

7d. Clearance at minor diameter.—A clearance shall be provided at the minor diameter of the nut by removing from the crest of the basic thread form an amount such as to provide a depth of thread not less than 53 to 75 percent (depending on the size), and not more than 83½ percent of the basic thread depth.

7e. Clearance at major diameter.—A clearance shall be provided at the major diameter of the nut by making the thread form such that the

width of flat shall be less than $\frac{1}{2} \times p$, but not less than $\frac{1}{24} \times p$.

7f. Thread series.—The present coarse-thread and fine-thread series are maintained, the coarse-thread series being the "United States standard" threads, supplemented in the sizes below one-fourth inch by sizes taken from the standard established by the American Society of Mechanical Engineers (ASME). The fine-thread series is composed of standards that have been found necessary and consists of sizes taken from the standards of the Society of Automotive Engineers (SAE) and the fine-thread series of the ASME.

7g. There are indicated in figure 3 the relations as specified herein for the American National form of thread for the minimum nut and

maximum screw, classes 2 and 3 fits.

² This standard, in substantially the same form, has been adopted by the American Standards Association. It is published, in part, as ASA Bl. 1—1935, Screw Threads, by the ASME, 29 West 39th Street, New York, N. Y.

AMERICAN NATIONAL COARSE-THREAD SERIES

8a. The American National coarse-thread series, as specified in table 1, is recommended for general use in engineering work, in machine construction where conditions are favorable to the use of bolts, screws, and other threaded components where quick and easy assembly of the parts is desired, and for all work where conditions do not require the use of fine-pitch threads. Limiting dimensions and tolerances for classes 1, 2, 3, and 4 fits are specified in table 15.

Table 1.—American National coarse-thread series

		Basic area of section at root of thread, $\frac{\pi K^2}{4}$	Sq. in. 0. 0022 0.0031 0.0041 0.0050 0.0067	$\begin{array}{c} .0075 \\ .0120 \\ .0145 \\ .0206 \end{array}$. 0269 . 0454 . 0678 . 0933	$\begin{array}{c} .\ 1620 \\ .\ 2018 \\ .\ 3020 \\ .\ 4193 \\ .\ 5510 \\ \end{array}$
		Helix angle at basic pitch diameter, s	Deg. Min. 4 22 4 22 4 25 4 45	4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	44 11 3 24 3 26 3 20 7 7	22 2 2 2 2 2 2 3 40 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6
	er.	Minimum width of flat at major di-ameter of nut, p/24	Inch 0. 00065 00074 00087 00104 00104	. 00130 . 00130 . 00174 . 00174	. 00208 . 00231 . 00260 . 00298	$\begin{array}{c} .00347 \\ .00379 \\ .00417 \\ .00463 \\ .00521 \\ \end{array}$
	Thread data	Basic width of flat, p/8	$egin{array}{l} Inch \\ 0.00195 \\ 0.00223 \\ 0.00260 \\ 0.00312 \\ 0.00312 \\ \end{array}$	$\begin{array}{c} .00391 \\ .00391 \\ .00521 \\ .00521 \end{array}$. 00625 . 00694 . 00781 . 00893 00962	$\begin{array}{c} .01042 \\ .01136 \\ .01250 \\ .01389 \\ .01562 \end{array}$
901 100		Depth of thread, h	$egin{array}{l} Inch \\ 0.01015 \\ 0.01160 \\ 0.01353 \\ 0.01624 \\ 0.01624 \\ \end{array}$. 02030 . 02030 . 02706 . 02706	. 03248 . 03608 . 04059 . 04639	. 05413 . 05905 . 06495 . 07217 . 08119
		Pitch, p	I_{nch} 0. 01562 0.01786 0.02083 0.02500 0.02500	. 03125 . 03125 . 04167 . 04167	. 05000 . 05556 . 06250 . 07143 . 07692	. 08333 . 09091 . 10000 . 111111
non amazona		Metric equivalent of major diameter	mm 1. 854 2. 184 2. 515 2. 845 3. 175	3. 505 4. 166 4. 826 5. 486	6. 350 7. 938 9. 525 11. 113 12. 700	14. 288 15. 875 19. 050 22. 225 25. 400
T almos tos		Minor di- ameter, K	Inches 0. 0527 . 0628 . 0719 . 0795	. 0974 . 1234 . 1359 . 1619	. 1850 . 2403 . 2938 . 3447 . 4001	. 4542 . 5069 . 6201 . 7307 . 8376
	Basic diameters	Pitch di- ameter, E	Inches 0. 0629 . 0744 . 0855 . 0958 . 1088	. 1177 . 1437 . 1629 . 1889	. 2175 . 2764 . 3344 . 3911 . 4500	. 5084 . 5660 . 6850 . 8028 . 9188
	Ba	Major di- ametcr, D	Inches 0. 073 . 086 . 099 . 112 . 125	. 138 . 164 . 190 . 216	. 2500 . 3125 . 3750 . 4375	. 5625 . 6250 . 7500 . 8750 1. 0000
		Threads per inch, n	64 48 40 40	32 32 44 44	20 18 16 14 13	11 11 10 9
	Identification	Sizes				
			22-44-75	6	7/16	1,44,74

. 6931 . 8898 1. 0541 1. 2938 1. 7441	2. 3001 3. 0212 3. 7161 4. 6194 5. 6209	6. 7205 7. 9183 9. 2143 10. 6084
31 24 11 15	11 55 57 46 36	29 16 11
80000	8	
. 00595 . 00595 . 00694 . 00694	. 00926 . 00926 . 01042 . 01042	. 01042 . 01042 . 01042 . 01042
. 01786 . 01786 . 02083 . 02083	. 02778 . 02778 . 03125 . 03125	. 03125 . 03125 . 03125 . 03125
. 09279 . 09279 . 10825 . 10825	. 14434 . 14434 . 16238 . 16238	. 16238 . 16238 . 16238 . 16238
. 14286 . 14286 . 16667 . 16667 . 20000	. 22222 . 22222 . 25000 . 25000	. 25000 . 25000 . 25000 . 25000
28. 575 31. 750 34. 925 38. 100 44. 450	50. 800 57. 150 63. 500 69. 850 76. 200	82. 550 88. 900 95. 250 101. 600
. 9394 1. 0644 1. 1585 1. 2835 1. 4902	1. 7113 1. 9613 2. 1752 2. 4252 2. 6752	2. 9252 3. 1752 3. 4252 3. 6752
1. 0322 1. 1572 1. 2667 1. 3917 1. 6201	1. 8557 2. 1057 2. 3376 2. 5876 2. 8376	3. 0876 3. 3376 3. 5876 3. 8376
1. 1250 1. 2500 1. 3750 1. 5000 1. 7500	2. 2500 2. 2500 2. 2500 3. 7500 3. 0000	3. 2500 3. 5000 3. 7500 4. 0000
2000	44444 %%	4444
25.7%	22/2 22/2 33/4	33/4

AMERICAN NATIONAL FINE-THREAD SERIES

8b. The American National fine-thread series as specified in table 2 is recommended for general use in automotive and aircraft work, and where special conditions require a fine thread. Limiting dimensions and tolerances for classes 1, 2, 3, and 4 fits are specified in table 16.

Table 2.—American National fine-thread series

	Basic area of section at root of thread, $\frac{\pi K^3}{4}$	$Sq.\ in.$ 0. 0015 0.0024 0.0034 0.0045	. 0072 . 0087 . 0128 . 0175	. 0326 . 0524 . 0809 . 1090	. 1888 . 2400 . 3513 . 4805 . 6464	. 8118 1. 0238 1. 2602 1. 5212
	Helix angle at basic pitch diameter,	Min. 23 57 45 43 51	244 228 23 23 24 25	52 40 11 15 57	55 443 34 22 22	25 16 9
	Hell at ba dia	Deg. 44.	000000	98989		
8	Basic width flat at major of flat, diameter of p/8 of nut, p/8 of nut, p/24	$\begin{array}{c} Inch \\ 0.00052 \\ 00058 \\ 00065 \\ 00074 \\ 00087 \end{array}$. 00095 . 00104 . 00116 . 00130 . 00149	. 00149 . 00174 . 00174 . 00208 . 00208	. 00231 . 00231 . 00260 . 00298 . 00298	. 00347 . 00347 . 00347 . 00347
Thread data	Basic width of flat, p/8	$\begin{array}{c} Inch \\ 0.\ 001156 \\ .\ 00174 \\ .\ 00195 \\ .\ 00223 \\ .\ 00260 \end{array}$. 00284 . 00312 . 00347 . 00391 . 00446	. 00446 . 00521 . 00521 . 00625 . 00625	. 00694 . 00694 . 00781 . 00893 . 00893	. 01042 . 01042 . 01042 . 01042
	Depth of thread,	1nch 0. 00812 0. 00902 0. 01015 0. 01160 0. 01353	. 01476 . 01624 . 01804 . 02030 . 02320	. 02320 . 02706 . 02706 . 03248 . 03248	. 03608 . 03608 . 04059 . 04639	$\begin{array}{c} .05413 \\ .05413 \\ .05413 \\ .05413 \\ .05413 \end{array}$
	Pitch,	$egin{array}{l} Inch \\ 0. \ 0.1250 \\ 0.01389 \\ 0.01562 \\ 0.01786 \\ 0.02083 \\ \end{array}$. 02273 . 02500 . 02778 . 03125 . 03571	. 03571 . 04167 . 04167 . 05000 . 05000	. 05556 . 05556 . 06250 . 07143 . 07143	. 08333 . 08333 . 08333
	Metric equivalent of major diameter	mm 1. 524 1. 854 2. 184 2. 515 2. 845	3. 175 3. 505 4. 166 4. 826 5. 486	6. 350 7. 938 9. 525 11. 113 12. 700	14. 288 15. 875 19. 050 22. 225 25. 400	28. 575 31. 750 34. 925 38. 100
SS	Minor di- ameter,	Inches 0. 0438 . 0550 . 0657 . 0758	. 0955 . 1055 . 1279 . 1494	. 2036 . 2584 . 3209 . 3725 . 4350	. 4903 . 5528 . 6688 . 7822 . 9072	1. 0167 1. 1417 1. 2667 1. 3917
Basic diameters	Pitch di- ameter, E	$\begin{array}{c} Inches \\ 0.\ 0519 \\ 0.0540 \\ 0.0759 \\ 0.0874 \\ 0.0985 \end{array}$. 1102 . 1218 . 1460 . 1697 . 1928	. 2268 . 2854 . 3479 . 4050	. 5264 . 5889 . 7094 . 8286 . 9536	1. 0709 1. 1959 1. 3209 1. 4459
В	$\begin{array}{c} \text{Major di-} \\ \text{ameter,} \\ D \end{array}$	Inches 0. 060 . 073 . 086 . 099 . 112	. 125 . 138 . 164 . 190 . 216	. 2500 . 3125 . 3750 . 4375	. 5625 . 6250 . 7500 . 8750 1. 0000	1. 1250 1. 2500 1. 3750 1. 5000
	Threads per inch, n	80 72 64 56 48	44 40 32 32 28	28 24 20 20 20	18 16 14 14	12 12 12 12
Identification	Sizes					
		0 1 2 3 4	6	3,74	288-	1138-1138-1138-1138-1138-1138-1138-1138

Table 15.—Limiting dimensions and tolerances, classes 1, 2, 3, and 4 fits, American National coarse-thread series

					Ma	chine screw	Machine screw number or nominal size	nominal s	ize			
tolomornoos		П	63	က	4	ro	9	∞	01	12	*	5/16
3						Th	Threads per inch	ch				
		64	56	48	40	40	32	32	24	24	20	18
Bolts and Screws [Max or diameter	ax 0 in	Inch 0. 0723 0671 0052	$\frac{Imch}{0.0852}$. 0796 0056	$1 \frac{Inch}{0.0981}$ 0.0981 0.0919 0.0062	$\frac{Inch}{0.\ 11110}$. $\frac{1042}{0068}$	Inch 0. 1240 . 1172 . 0068	Inch 0. 1369 . 1293 . 0076	Inch 0. 1629 . 1553 . 0076	0. 1887 1795 . 0092	$\begin{array}{c} Inch \\ 0.\ 2147 \\ .\ 2055 \\ .\ 0092 \end{array}$	Inch 0. 2485 . 2383 . 0102	$1mch \\ 0.3109 \\ 2995 \\ 0.0114$
diameter_{Min{Tol_	ax	. 0730 . 0692 . 0038	. 0860 . 0820 . 0040	. 0990 . 0946 . 0044	. 1120 . 1072 . 0048	. 1250 . 1202 . 0048	. 1326 . 1326 . 0054	. 1640 . 1586 . 0054	. 1900 . 1834 . 0066	. 2160 . 2094 . 0066	. 2500 . 2428 . 0072	. 3125 . 3043 . 0082
(threaded Max hot-rolled Min. Tol.	ax	. 0730 . 0678 . 0052	. 0860 . 0804 . 0056	. 0990 . 0928 . 0062	. 1120 . 1052 . 0068	. 1250 . 1182 . 0068	. 1380 . 1304 . 0076	. 1640 . 1564 . 0076	. 1900 . 1808 . 0092	. 2160 . 2068 . 0092	. 2500 . 2398 . 0102	. 3125 . 3011 . 0114
Ma	Max.1	. 0531	. 0633	. 0725	. 0803	. 0933	. 0986	. 1246	. 1376	. 1636	. 1872	. 2427
diameter Max.	-	. 0538	. 0641	. 0734	. 0813	. 0943	. 0997	. 1257	. 1389	. 1649	. 1887	. 2443
Min. Tol.	ax	. 0622 . 0596 . 0026	. 0736 . 0708 . 0028	$\begin{array}{c} .\ 0846 \\ .\ 0815 \\ .\ 0031 \end{array}$. 0948 . 0914 . 0034	. 1078 . 1044 . 0034	. 1166 . 1128 . 0038	. 1426 . 1388 . 0038	. 1616 . 1570 . 0046	. 1876 . 1830 . 0046	. 2160 . 2109 . 0051	. 2748 . 2691 . 0057
{Max. Min. Tol.	ax	. 0629 . 0610 . 0019	. 0744 . 0724 . 0020	. 0855 . 0833 . 0022	. 0958 . 0934 . 0024	. 1088 . 1064 . 0024	11177 . 1150	1437 1410 0027	$\begin{array}{ c c c c c c c c c c c c c c c c c c c$. 1889 . 1856 . 0033	2175 2139 0036	. 2764 . 2723 . 0041

. 2764 . 2734 . 0030	. 2767 . 2752 . 0015		0. 3125	. 2630 . 2524 . 0106	. 2764	. 2821	. 2805	. 2794	. 2779
. 2175 . 2149 . 0026	. 2178 . 2165 . 0013		0. 2500	. 2060 . 1959 . 0101	. 2175	. 2226	. 2211	. 2201	. 2188
. 1889			0. 2160	. 1801 . 1709 . 0092	. 1889	. 1935	. 1922	. 1913	, I I I I I I I I I I I I I I I I I I I
. 1629 . 1605 . 0024	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		0. 1900	. 1559 . 1449 . 0110	. 1629	. 1675	. 1662	. 1653	
. 1437 . 1418 . 0019	1 1 1 1 1 1 1 1 1 1	,	0. 1640	$\begin{array}{c} .1384 \\ .1302 \\ .0082 \end{array}$. 1437	. 1475	. 1464	. 1456	1 1 1 1 1 1 1 1 1 1 1 1
. 1158			0. 1380	$\begin{array}{c} .\ 1145 \\ .\ 1042 \\ .\ 0103 \end{array}$. 1177	. 1215	. 1204	. 1196	
. 1088 . 1071			0. 1250	$\begin{array}{c} .\ 1062 \\ .\ 0979 \\ .\ 0083 \end{array}$. 1088	$\frac{1122}{0034}$. 1112	. 1105	
. 0958 . 0941 . 0017			0.1120	$\begin{array}{c} .\ 0938 \\ .\ 0849 \\ .\ 0089 \end{array}$. 0958	. 0992	. 0982	. 0975	
. 0855 . 0839 . 0016			0.0990	. 0841 . 0764 . 0077	. 0855	. 0886	. 0877	. 0871	1
. 0744 . 0729 . 0015			0.0860	. 0737 . 0667 . 0070	. 0744	. 0772	. 0764	. 0759	
. 0629 . 0615 . 0014			0.0730	0623 0561 0062	. 0629	. 0655	. 0648	. 0643	
${f Min_{} \atop Tol_{}}$	Max Min Tol		- Min.²	Min Tol	-Min	{Max.4 {Tol	{Max.4	{Max.4 {Tol	${ m Max.}^4$ ${ m Tol}$
	Class 4, pitch diameter	NUTS AND TAPPED HOLES	© Classes 1, 2, 3, and 4, major diameter_Min.² 0. 0730	Classes 1, 2, 3, and 4, minor diameter	Classes 1, 2, 3, and 4, pitch diameter.	Class 1, pitch diameter $\left\{ \right.$	Class 2, pitch diameter	Class 3, pitch diameter	Class 4, pitch diameter

See footnotes at end of table.

Table 15.—Limiting dimensions and tolerances, classes 1, 2, 3, and 4 fits, American National coarse-thread series—Continued

			2010		aas	coroco .	L cop L		J ()
1. 2667 1. 2596 0. 0071	1. 2676 1. 2640 0. 0036		1, 3750	1. 2126 1. 1946 0. 0180	1. 2667	$\frac{1.2812}{0.0145}$	$\frac{1.2768}{0.0101}$	1. 2738 0. 0071	1. 2703 0. 0036
1. 1572 1. 1513 0. 0059	1. 1580 1. 1550 0. 0030		1, 2500	1. 1108 1. 0954 0. 0154	1. 1572	1. 1696 0. 0124	1. 1657 0. 0085	1. 1631 0. 0059	1. 1602 0. 0030
1. 0322 1. 0263 0. 0059	1. 0330 1. 0300 0. 0030		1, 1250	$0.9858 \\ .9704 \\ .0154$	1, 0322	1. 0446 0. 0124	1. 0407 0. 0085	1. 0381 0. 0059	$\begin{array}{c} 1.\ 0352 \\ 0.\ 0030 \end{array}$
$\begin{array}{c} .9188 \\ .9134 \\ .0054 \end{array}$. 9195 . 9168 . 0027		1. 0000	$0.8795 \\ .8647 \\ .0148$. 9188	. 9299	. 9264	. 9242	. 9215
. 8028 . 7979 . 0049	. 8034 . 8010 . 0024		0.8750	. 7689 . 7547 . 0142	. 8028	. 8128	8088	. 8077	. 8052 . 0024
. 6850 . 6805 . 0045	. 6856 . 6833 . 0023	-	0.7500	. 6553 . 6417 . 0136	. 6850	. 6942	. 6914	. 6895	. 6873
. 5660 . 5618 . 0042	. 5665 . 5644 . 0021		0. 6250	. 5397 . 5266 . 0131	. 5660	. 5745	. 5719	. 5702	. 5681
. 5084 . 5044 . 0040	. 5089 . 5069 . 0020		0. 5625	. 4850 . 4723 . 0127	. 5084	. 5163	. 5140	. 5124	. 5104
. 4500 4463 . 0037	. 4504 . 4485 . 0019		0. 5000	. 4290 . 4167 . 0123	. 4500	. 4574	. 4552	. 4537	. 4519
. 3911 . 3875 . 0036	. 3915 . 3897 . 0018		0. 4375	. 3721 . 3602 . 0119	3911	. 3981	. 3960	. 3947	. 3929 . 0018
. 3344 . 3312 . 0032	. 3348 . 3332 . 0016		0.3750	. 3184 . 3073 . 0111	. 3344	. 3407	. 3389	. 3376	. 3360
Max Min Tol	{ Max { Min Tol	Ω	diameter Min ²	Max ³ Min Tol	r- Min	-{Max 4 Tol	-{Max 4	-{Max 4	-{Max 4
Class 3, pitch diameter	Class 4, pitch diameter	NUTS AND TAPPED HOLES	Classes 1, 2, 3, and 4, major	Classes 1, 2, 3, and 4, minor diameter	Classes 1, 2, 3, and 4, pitch diameter. Min	Class 1, pitch diameter	Class 2, pitch diameter	Class 3, pitch diameter	Class 4, pitch diameter

See footnotes at end of table.

Table 15.—Limiting dimensions and tolerances, classes 1, 2, 3, and 4 fits, American National coarse-thread series—Continued

						Size					
Dimonolone and tolonouses	11/2	134	64	21/4	21/2	23/4	en	314	31/2	334	4
Untensions and totalices					Th	Threads per inch	ą.				
	9	2	41/2	43/2	4	₩.	₹	₩.	4	4	4
Bolts and Screws Class 1, major diam- eter	Inches 1. 4956 1. 4666 0. 0290	Inches 1. 7448 1. 7110 0. 0338	Inches 1. 9943 1. 9575 0. 0368	Inches 2. 2443 2. 2075 0. 0368	Inches 2. 4936 2. 4528 0. 0408	Inches 2. 7436 2. 7028 0. 0408	Inches 2. 9936 2. 9528 0. 0408	Inches 3. 2436 3. 2028 0. 0408	Inches 3. 4936 3. 4528 0. 0408	Inches 3. 7436 3. 7028 0. 0408	Inches 3. 9936 3. 9528 0. 0408
Classes 2, 3, and 4, Minmajor diameter Tol	1. 5000 1. 4798 0. 0202	1. 7500 1. 7268 0. 0232	2. 0000 1. 9746 0. 0254	2. 2500 2. 2246 0. 0254	2. 5000 2. 4720 0. 0280	2. 7500 2. 7220 0. 0280	3. 0000 2. 9720 0. 0280	3. 2500 3. 2220 0. 0280	3. 5000 3. 4720 0. 0280	3. 7500 3. 7220 0. 0280	4. 0000 3. 9720 0. 0280
Class 2, major diam- (Max eter (threaded parts Min of unfinished, hot- Tol rolled material)	1. 5000 1. 4710 0. 0290	1. 7500 1. 7162 0. 0338	2. 0000 1. 9632 0. 0368	2, 2500 2, 2132 0, 0368	2. 5000 2. 4592 0. 0408	2. 7500 2. 7092 0. 0408	3. 0000 2. 9592 0. 0408	3. 2500 3. 2092 0. 0408	3. 5000 3. 4592 0. 0408	3. 7500 3. 7092 0. 0408	4, 0000 3, 9592 0, 0408
Class 1 minor diameter_Max.¹_Classes 2, 3, and 4, minor diameterMax.¹	1. 2911	1. 4994	1. 7217	1. 9717	2. 1869	2, 4369	2. 6869	2. 9369 2. 9433	3. 1869 3. 1933	3, 4369	3, 6869
Class 1, pitch diameter. $Min_{}$ Tol	1. 3873 1. 3728 0. 0145	1. 6149 1. 5980 0. 0169	1. 8500 1. 8316 0. 0184	2. 1000 2. 0816 0. 0184	2. 3312 2. 3108 0. 0204	2. 5812 2. 5608 0. 0204	2. 8312 2. 8108 0. 0204	3. 0812 3. 0608 0. 0204	3. 3312 3. 3108 0. 0204	3. 5812 3. 5608 0. 0204	3. 8312 3. 8108 0. 0204
Class 2, pitch diameter \(\text{Min} \) \(\text{Inl} \)	1. 3917 1. 3816 0. 0101	1. 6201 1. 6085 0. 0116	1. 8557 1. 8430 0. 0127	2. 1057 2. 0930 0. 0127	2. 3376 2. 3236 0. 0140	2. 5876 2. 5736 0. 0140	2. 8376 2. 8236 0. 0140	3. 0876 3. 0736 0. 0140	3. 3376 3. 3236 0. 0140	3. 5876 3. 5736 0. 0140	3. 8376 3. 8236 0. 0140

			.0000				-T			
3. 8376 3. 8279 0. 0097	3, 8389 3, 8341 0, 0048		4. 0000	3. 7564 3. 7294 0. 0270	3. 8376	3. 8580 0. 0204	3. 8516 0. 0140	3. 8473 0. 0097	3. 8424 0. 0048	minimum asic thread
3. 5876 3. 5779 0. 0097	3. 5889 3. 5841 0. 0048		3. 7500	3. 5064 3. 4794 0. 0270	3. 5876	3. 6080 0. 0204	3. 6016 0. 0140	3. 5973 0. 0097	3. 5924 0. 0048	root. The
3. 3376 3. 3279 0. 0097	3. 3389 3. 3341 0. 0048		3. 5000	3. 2564 3. 2294 0. 0270	3. 3376	3. 3580 0. 0204	3. 3516 0. 0140	3. 3473 0. 0097	3. 3424 0. 0048	igh crest and
3. 0876 3. 0779 0. 0097	3. 0889 3. 0841 0. 0048		3, 2500	3. 0064 2. 9794 0. 0270	3. 0876	3. 1080 0. 0204	3. 1016 0. 0140	3. 0973 0. 0097	3. 0924 0. 0048	ter line throu
2. 8376 2. 8279 0. 0097	2. 8389 2. 8341 0. 0048		3. 0000	2. 7564 2. 7294 0. 0270	2. 8376	2. 8580 0. 0204	2. 8516 0. 0140	2. 8473 0. 0097	2. 8424 0. 0048	c with a cen
2. 5876 2. 5779 0. 0097	2. 5889 2. 5841 0. 0048		2. 7500	2. 5064 2. 4794 0. 0270	2. 5876	2. 6080 0. 0204	2. 6016 0. 0140	2. 5973 0. 0097	2. 5924 0. 0048	worn tool ar
2. 3376 2. 3279 0. 0097	2. 3389 2. 3341 0. 0048	,	2. 5000	2. 2564 2. 2294 0. 0270	2. 3376	2. 3580 0. 0204	2. 3516 0. 0140	2. 3473 0. 0097	2. 3424 0. 0048	ection of the
2. 1057 2. 0968 0. 0089	2. 1068 2. 1024 0. 0044		2. 2500	2. 0335 2. 0094 0. 0241	2. 1057	2. 1241 0. 0184	2. 1184 0. 0127	2. 1146 0. 0089	2. 1101 0. 0044	to the inters
1. 8557 1. 8468 0. 0089	1. 8568 1. 8524 0. 0044		2. 0000	1. 7835 1. 7594 0. 0241	1. 8557	1. 8741 0. 0184	1. 8684 0. 0127	1. 8646 0. 0089	1. 8601 0. 0044	v are figured
1. 6201 1. 6119 0. 0082	1. 6211 1. 6170 0. 0041		1. 7500	1. 5551 1. 5335 0. 0216	1. 6201	1. 6370 0. 0169	1. 6317 0. 0116	1. 6283 0. 0082	1. 6242 0. 0041	aximum minor diameter of the screw are figured to the intersection of the worn tool are with a center line through crest and root. The minimum he that corresponding to a flat at the minor diameter of the minimum screw cental to 18.82, and may be determined by subtracting the basic thread
1. 3917 1. 3846 0. 0071	1. 3926 1. 3890 0. 0036		1. 5000	1. 3376 1. 3196 0. 0180	1. 3917	1. 4062 0. 0145	1. 4018 0. 0101	1. 3988 0. 0071	1. 3953 0. 0036	inor diamete
Class 3, pitch diameter-\{\text{Min}\text{Tol}	Class 4, pitch diameter \(\text{Min} \)	NUTS AND TAPPED HOLES	Classes 1, 2, 3, and 4, major diameterMin.²	Classes 1, 2, 3, and 4, Max.*- minor diameter[Tol	Classes 1, 2, 3, and 4, pitch diameterMin	Class 1, pitch diameter ${\rm Max.}^{4}$	Class 2, pitch diameter $\{\text{Max.}^4$	Class 3, pitch diameter $\{\text{Nax.}^{4}$.	Class 4, pitch diameter-\{\text{Max.4}\Tol	Dimensions given for the maximum minor diameter of the screw are figured to the intersection of the worn tool are with a center line through crest and root.

minor diameter of the screw shall be that corresponding to a flat at the minor diameter of the minimum pitch diameter of the serve depth, b(or 0.6495 p), from the minimum pitch diameter of the serve depth, b(or 0.6495 p), from the minimum pitch diameter of the serve depth, b(or 0.6495 p), from the minimum pitch diameter of the nut correspond to a flat at the profile at the major diameter produced by a worn tool must not fall below the basic outline. The maximum major diameter of the nut shall be that corresponding to a flat at the major diameter of the must not a present of the maximum pitch diameter of the nut.

**Jenson 1.52 **A (or 0.7939 p) to the maximum pitch diameter of the nut.

**Jenson 1.52 **A (or 0.7939 p) to the maximum pitch diameter of the nut.

**Jenson 1.52 **A (or 0.7939 p) to the maximum pitch diameter of the nut.

**Jenson 1.52 **A (or 0.7939 p) to the maximum pitch diameter of the nut.

**Jenson 1.52 **A (or 0.7939 p) to the maximum pitch diameters of nuts.

**Jenson 1.52 **A (or 0.7939 p) to the maximum minor diameters of nuts.

**These dimensions are the minimum metal or "not go" size. The "go" or basic size is the one that should be placed on the component drawing with the tolerance.

Table 16.—Limiting dimensions and tolerances, classes 1, 2, 3, and 4 fits, American National fine-thread series

					2	fachine sc	rew num	Machine screw number or nominal size	ninal size					
i	0	1	m	m	4	10	9	00	01	12	*	5/16	3%	7.16
Dimensions and tolerances		-					Threads per inch	er inch						
	80	72	64	56	48	44	40	36	32	88	28	24	24	20
Bolts and Screws Class I, major diam{Min. Tol	Inch 0. 0593 0 0545	Inch 0723 0673 0050	0. 0853 0. 0801 0. 0052	Inch 0. 0982 0. 0926 0056	Inch 11111 1049 . 0062	Inch 1. 1241 0. 1. 1177 . 0064	Inch 1. 1370 1. 1302 1. 0068	Inch 1370 0. 1629 0. 1302 . 1557 . 0068 . 0072	$\begin{bmatrix} Inch \\ 1889 \\ 1813 \\ 0076 \end{bmatrix}$	Inch 0. 2148 0. 2062 0. 0086	Inch 0. 2488 0. 2402 0086	Inch 0. 3112 0. 3020 . 0092	$\begin{bmatrix} Inch \\ 0.3737 \\ .3645 \\ .0092 \end{bmatrix}$.	$^{Inch}_{0.4360}$. $^{4260}_{0.0102}$.
Classes 2, 3, and 4, major $\begin{cases} Max. \\ diam. \end{cases}$. 0730 . 0694 . 0036	. 0860 . 0822 . 0038	. 0990 . 0950 . 0040	$\begin{array}{c} .1120 \\ .1076 \\ .0044 \end{array}$	$\begin{array}{c} .1250 \\ .1204 \\ .0046 \end{array}$. 1380 . 1332 . 0048	$\begin{array}{c} .1640 \\ .1590 \\ .0050 \end{array}$	$\begin{array}{c} .1900 \\ .1846 \\ .0054 \end{array}$. 2160 . 2098 . 0062	. 2500 . 2438 . 0062	. 3125 . 3059 . 0066	$\begin{array}{c} .3750 \\ .3684 \\ .0066 \end{array}$. 4375 . 4303 . 0072
Class 1, minor diamMax. ¹ Classes 2, 3, and 4, minor diam Max. ¹	0440	. 0553	. 0668	. 0763	. 0855	. 0962	. 1063	. 1288	. 1506	. 1710	. 2050	. 2601	. 3226	. 3747
pitch diam			. 0752 . 0726 . 0026	. 0866	. 0976 . 0945 . 0031	$\begin{array}{c} 1093 \\ 1061 \\ 0032 \end{array}$	$\begin{array}{c} 1208 \\ 1174 \\ 0034 \end{array}$. 1449 . 1413 . 0036	$\begin{array}{c} 1686 \\ 1648 \\ 0038 \end{array}$. 1916 . 1873 . 0043	. 2256 . 2213 . 0043	. 2841 . 2795 . 0046	. 3466 . 3420 . 0046	. 4035 . 3984 . 0051
Class 2, pitch diam{ Max. Trib		. 0640 . 0622 . 0018	0759 0740 0019	. 0874 . 0854 . 0020	. 0985 . 0963 . 0022	. 1102 . 1079 . 0023	$\begin{array}{c} .1218 \\ .1194 \\ .0024 \end{array}$. 1460 . 1435 . 0025	$\begin{array}{c} .1697 \\ .1670 \\ .0027 \end{array}$	$\frac{1928}{1897}$. 2268 . 2237 . 0031	. 2854 . 2821 . 0033	. 3479 . 3446 . 0033	$\begin{array}{c} .4050 \\ .4014 \\ .0036 \end{array}$
Class 3, pitch diam $\begin{cases} Max. \\ Min. \end{cases}$ Class 4, pitch diam $\begin{cases} Max. \\ Min. \end{cases}$. 0640	. 0759	. 0874 . 0859 . 0015	. 0985	1102	1218	. 1460	. 1697	. 1928	. 2268 . 2246 . 0022 . 2270 . 2259	2854 2830 0024 2857 2857 0012	3479 3455 0024 3482 3482 3470	. 4050 . 4024 . 0026 . 4053 . 4040 . 0013

· 10	0 42	0	==	99	99	က္ကေက ၂
5	. 4373 . 3906 . 3834 . 0072	. 4050	. 4101	. 3086	$\frac{4076}{0026}$. 4063
1	. 3364 . 3299 . 0065	. 3479	$\frac{3525}{0046}$. 3512	. 3503	. 3491
0	. 2739 . 2674 . 0065	. 2854	. 2900	. 2887	$\frac{2878}{0024}$. 2866
2	. 2173 . 2113 . 0060		. 2311	. 2299	. 2290	. 2279
- 0	. 2160 . 1835 . 1773 . 0062	. 1928	. 1971	. 1959	. 1950	
00	. 1350 . 1040 . 1900 . 2100 . 2250 . 3125 . 3750 . 1179 . 1402 . 1624 . 1835 . 2173 . 2739 . 3364 . 1109 . 1339 . 1562 . 1773 . 2113 . 2674 . 3299 . 0070 . 0063 . 0062 . 0062 . 0066 . 0065 . 0065	. 1697	. 1735	. 1724	. 1716	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
9	. 1402 . 1339 . 0063	. 1218 . 1460	. 1496	. 1485	. 1235 . 1478 . 0017 . 0018	
000	. 1380 . 1179 . 1109	. 1218	. 1252	. 1242	. 1235	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
C C	. 1250 . 1068 . 1004 . 0064		. 1134	$\frac{1125}{0023}$. 1118	-
ç	. 0856 . 0960 . 1068 . 0797 . 0894 . 1004 . 0059 . 0066 . 0064	. 0985	. 1016	. 1007	. 1001	
			. 0902	. 0894	. 0889 . 0015	1 1
0	0.0860 0.0746 0.0691 0.0055	. 0759	. 0785	. 00778	. 0014	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
9	. 0730 . 0634 . 0580 . 0054	. 0640	. 0665	. 0658	. 0653	
		. 0519	. 0543	. 0536	. 0532	1 1 1 1 1 1 1 1 1 1
Holes	Min. Z. Min. Min. Tol	Min 0519	{Tol0543	-{Max.40536	{Max.40532	{Max.4
NUTS AND TAPPED I	major diam	Classes 1, 2, 3, and 4, pitch diam	Class 1, pitch diam	Class 2, pitch diam	Class 3, pitch diam	Class 4, pitch diam
N U	majo Classes minol	Classes	Class 1,	Class 2,	Class 3,	Class 4,

See footnotes at end of table.

Table 16.—Limiting dimensions and tolerances, classes 1, 2, 3, and 4 fits, American National fine-thread series—Continued

					Size	89				
Dimensions and toleranes	22	976			22	1	11/8	11/4	13/8	11/2
LIMELISIONS AND COST BLOCKS					Threads per inch	per inch				
	20	18	18	16	14	14	12	12	12	12
Bolts and Screws (Max	Inch 0. 4985 4883 . 0102	Inch 0. 5609 . 5495 . 0114	Inch 0. 6234 . 6120 . 0114	Inch 0. 7482 . 7356 . 0126	Inch 0. 8729 . 8589 . 0140	Inch 0. 9979 . 9839 . 0140	Inches 1. 1226 1. 1068 0. 0158	Inches 1. 2476 1. 2318 0. 0158	Inches 1. 3726 1. 3568 0. 0158	Inches 1. 4976 1. 4818 0. 0158
Classes 2, 3, and 4, major diam{ Min	. 5000 . 4928 . 0072	. 5625 . 5543 . 0082	. 6250 . 6168 . 0082	. 7500 . 7410 . 0090	. 8750 . 8652 . 0098	1. 0000 0. 9902 . 0098	1. 1250 1. 1138 0. 0112	1. 2500 1. 2388 0. 0112	1. 3750 1. 3638 0. 0112	1. 5000 1. 4888 0. 0112
Class 1, minor diamMax.¹Classes 2, 3, and 4, minor diam Max.¹	. 4387	. 4927	. 5552	. 6715 . 6733	. 7853	. 9103	1. 0204 1. 0228	1. 1454 1. 1478	1. 2704 1. 2728	1. 3954 1. 3978
Class 1, pitch diam	. 4660 . 4609 . 0051	$\begin{array}{c} .5248 \\ .5191 \\ .0057 \end{array}$. 5873 . 5816 . 0057	. 7076 . 7013 . 0063	. 8265 . 8195 . 0070	. 9515 . 9445 . 0070	1. 0685 1. 0606 0. 0079	1. 1935 1. 1856 0. 0079	1. 3185 1. 3106 0. 0079	1. 4435 1. 4356 0. 0079
Class 2, pitch diam\Min\Tol	. 4675 . 4639 . 0036	$\begin{array}{c} .5264 \\ .5223 \\ .0041 \end{array}$. 5889 . 5848 . 0041	. 7094 . 7049 . 0045	. 8286 . 8237 . 0049	. 9536 . 9487 . 0049	1. 0709 1. 0653 0. 0056	1. 1959 1. 1903 0. 0056	1. 3209 1. 3153 0. 0056	1. 4459 1. 4403 0. 0056
Class 3, pitch diam\ Min	. 4675 4649 . 0026	. 5264 . 5234 . 0030	. 5889 . 5859 . 0030	. 7094 . 7062 . 0032	. 8286 . 8250 . 0036	. 9536 . 9500 . 0036	1. 0709 1. 0669 0. 0040	1. 1959 1. 1919 0. 0040	1. 3209 1. 3169 0. 0040	1. 4459 1. 4419 0. 0040
Class 4, pitch diam{Min	. 4678 . 4665 . 0013	. 5267 . 5252 . 0015	. 5892 . 5877 . 0015	. 7098 . 7082 . 0016	. 8290 . 8272 . 0018	9540	1. 0714 1. 0694 0. 0020	1. 1964 1. 1944 0. 0020	1. 3214 1. 3194 0. 0020	1. 4464 1. 4444 0. 0020

Classes 1, 2, 3, and 4, major diam Min. 2. Classes 1, 2, 3, and 4, major diam Min. 2. Classes 1, 2, 3, and 4, major diam Min. 2. Classes 1, 2, 3, and 4, major diam Min. 2. Classes 1, 2, 3, and 4, major diam Min. 2. Classes 1, 2, 3, and 4, minor diam $\begin{cases} Max.^3 - 4531 \\ Min 4459 \\ Min 60072 \end{cases}$ Classes 1, 2, 3, and 4, minor diam $\begin{cases} Max.^4 - 4531 \\ Min 6072 \\ Min 6076 \end{cases}$ Classes 1, 2, 3, and 4, minor diam $\begin{cases} Max.^4 - 4531 \\ Min 6072 \\ Min 6076 \end{cases}$ Classes 1, 2, 3, and 4, pitch diam. Min. 2. Classe 1, 2, 3, and 4, pitch diam. Min. 2. Classe 1, 2, 3, and 4, pitch diam. Min. 2. Classe 1, 2, 3, and 4, pitch diam. Min. 2. Class 1, pitch diam. Min. 2. Class 2, pitch diam. Min. 2. Class 3, pitch diam. Min. 2. Class 4, pitch diam. Min. 2. Class 4, pitch diam. Min. 2. Class 5, pitch diam. Min. 2. Class 6, pitch diam. Min. 2. Class 7, 7977 Class 8, 7977 Class 7, 7977 Class 8, 7977 Class 8, 1, 1688 Class 9, 7094 Class
$\begin{array}{cccccccccccccccccccccccccccccccccccc$
Holes diam Min.2 5000 . 5625 . diam Min.2 4531 . 5100 . diam. $\begin{cases} Max^4 - 4531 & 4726 & 5024 \\ Tol 4675 & 5024 \\ Tol 4675 & 5264 \\ Max 4 4726 & 5321 \\ 0057 & 0057 \\ \begin{cases} Max 4 4726 & 5321 \\ 0057 & 0057 \\ 0026 & 0036 \\ \begin{cases} Max 4 4701 & 5.305 \\ 0026 & 0030 \\ 0026 & 0030 \\ 0026 & 0030 \\ 0026 & 0030 \\ 0026 & 0030 \\ 0026 & 0030 \\ 0026 & 0030 \\ 0026 & 0030 \\ 0026 & 0030 \\ 0026 & 0030 \\ 0030 & 0015 \\ 0015 & 0015 \\$
diam. Min. 2 5000
Holes : diam Min.2 diam Min diam Min [Max.4 [Max.4 Tol [Max.4 Tol [Max.4 Tol
Hole diam diam
500943°—43——4

1 Dimensions given for the maximum minor diameter of the screw are figured to the intersection of the worn tool are with a center line through crest and root. The minimum halor diameter of the screw shall be that corresponding to a flat at the minor diameter of the minimum pitch diameter of the minimum pitch diameter of the minimum pitch diameter of the new of the minimum major diameter of the new of the minimum major diameter of the new of the paste of the minimum major diameter of the new of the minimum major diameter of the new of the new of the minimum major diameter of the new of the new of the minimum major diameter of the new of the new of the minimum major diameter of the new of the new of the minimum major diameter of the new of the new of the minimum major diameter of the new of the new of the minimum major diameter of the new of the new of the minimum major diameter of the new of the new of the minimum minor diameter of the new of the minimum minor diameter of the minimum metal or "not go" size. The "go" or basic size is the one that should be placed on the component drawing with the tolerance.

UNIFORM-PITCH SCREW-THREAD SERIES FOR HIGH-PRESSURE FASTENINGS, BOILER APPLICATIONS, MACHINERY COMPONENTS, ETC.3

FORM OF THREAD

9. The American National form of thread profile as specified in paragraphs 7 to 7f shall be used.

THREAD SERIES

9a. Where special threads are required, it is sometimes essential to select a certain pitch as standard for a range of sizes. Also, in general practice, where the pitch of a special thread is optional, the uniform use of a selected pitch is advantageous. For such applications 8, 12, and 16 threads per inch are widely used.

AMERICAN NATIONAL 8-PITCH-THREAD SERIES

9b. In table 26 are specified the nominal sizes and basic dimensions of the "American National 8-pitch-thread series." Limiting dimensions and tolerances for classes 2 and 3 fits are specified in table 29.

Bolts for high-pressure pipe flanges, cylinder-head studs, and similar fastenings against pressure require that an initial tension be set up in the fastening, by elastic deformation of the fastening and the components held together, such that the joint will not open up when the steam or other pressure is applied. To secure a proper initial tension it is not practicable that the pitch should increase with the diameter of the thread, as the torque required to assemble the fastening would be excessive. Accordingly, for such purposes the 8-pitch thread has come into general use.

AMERICAN NATIONAL 12-PITCH-THREAD SERIES

9c. The nominal sizes and basic dimensions of the "American National 12-pitch-thread series" are specified in table 27. Limiting dimensions and tolerances for classes 2 and 3 fits are specified in table 30.

Sizes of 12-pitch threads from one-half inch to and including one and three-fourths inches are used in boiler practice, which requires that worn stud holes be retapped with a tap of the next larger size, the increment being one-sixteenth inch throughout most of the range. Die-head chasers for sizes up to 3 inches are stocked by manufacturers.

The 12-pitch threads are also widely used in machine construction, as for thin nuts on shafts and sleeves. From the standpoints of good design and simplification of practice, it is desirable to limit shoulder diameters to %-inch steps. The 12 pitch is the coarsest in general use that will permit a threaded collar which screws onto a threaded shoulder to slip over a shaft, the difference in diameter between shoulder and shaft being one-eighth inch.

³ This standard, in substantially the same form, has been adopted by the American Standards Association. It is published as ASA B1.1—1935 "Screw Threads" by the ASME, 29 West 39th St., New York, N. Y. ⁴ See U. S. Department of Commerce Simplified Practice Recommendation R51–29, Die Head Chasers.

AMERICAN NATIONAL 16-PITCH-THREAD SERIES

9d. The nominal sizes and basic dimensions of the "American National 16-pitch-thread series" are specified in table 28. Limiting dimensions and tolerances for classes 2 and 3 fits are specified in table 31.

The 16-pitch series is a uniform pitch series for such applications as require a relatively fine thread. It is intended primarily for use on threaded adjusting collars and bearing-retaining nuts.

Table 26.—American National 8-pitch thread series

[Pitch, p=0.12500 inch; depth of thread, h=0.08119 inch; basic width of flat, p/8=0.01562 inch; minimum width of flat at major diameter of nut, p/24=0.00521 inch.]

	WIG	I of hat at i	major diame	ter or nut, p	24=0.00521 in	CH.]	
Identific	ation	В	asic diamete	rs		Thread data	
Sizes	Threads per inch	$egin{array}{c} ext{Major} \ ext{diameter,} \ ext{\it D} \end{array}$	$\begin{array}{c} \text{Pitch} \\ \text{diameter,} \\ E \end{array}$	Minor diameter, K	Metric equivalent of major diameter	Helix angle at basic pitch diameter,	Basic area of section at root of thread, $\frac{\pi K^2}{4}$
Inches 11	8 8 8 8	Inches 1. 0000 1. 1250 1. 2500 1. 3750 1. 5000	Inches 0. 9188 1. 0438 1. 1688 1. 2938 1. 4188	Inches 0. 8376 . 9626 1. 0876 1. 2126 1. 3376	25. 400 28. 575 31. 750 34. 925 38. 100	$\begin{array}{cccc} & deg & min \\ 2 & & 29 \\ 2 & & 11 \\ 1 & & 57 \\ 1 & & 46 \\ 1 & & 36 \end{array}$	Square inches 0. 5510 . 7277 . 9290 1. 1548 1. 4052
15% 134 17% 2 2½	8 8 8 8	1. 6250 1. 7500 1. 8750 2. 0000 2. 1250	1. 5438 1. 6688 1. 7938 1. 9188 2. 0438	1. 4626 1. 5876 1. 7126 1. 8376 1. 9626	41. 275 44. 450 47. 625 50. 800 53. 975	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	1. 6801 1. 9796 2. 3036 2. 6521 3. 0252
2½	8 8 8 8	2. 2500 2. 5000 2. 7500 3. 0000 3. 2500	2. 1688 2. 4188 2. 6688 2. 9188 3. 1688	2. 0876 2. 3376 2. 5876 2. 8376 3. 0876	57. 150 63. 500 69. 850 76. 200 82. 550	$\begin{array}{ccc} 1 & 3 \\ 0 & 57 \\ 0 & 51 \\ 0 & 47 \\ 0 & 43 \end{array}$	3. 4228 4. 2917 5. 2588 6. 3240 7. 4874
3½	8 8 8 8	3. 5000 3. 7500 4. 0000 4. 2500 4. 5000	3. 4188 3. 6688 3. 9188 4. 1688 4. 4188	3. 3376 3. 5876 3. 8376 4. 0876 4. 3376	88. 900 95. 250 101. 600 107. 950 114. 300	$\begin{array}{ccc} 0 & 40 \\ 0 & 37 \\ 0 & 35 \\ 0 & 33 \\ 0 & 31 \end{array}$	8. 7490 10. 1088 11. 5667 13. 1228 14. 7771
4 ³ / ₄	8 8 8 8	4. 7500 5. 0000 5. 2500 5. 5000 5. 7500	4. 6688 4. 9188 5. 1688 5. 4188 5. 6688	4. 5876 4. 8376 5. 0876 5. 3376 5. 5876	120. 650 127. 000 133. 350 139. 700 146. 050	$\begin{array}{ccc} 0 & 29 \\ 0 & 28 \\ 0 & 26 \\ 0 & 25 \\ 0 & 24 \end{array}$	16. 5295 18. 3802 20. 3290 22. 3760 24. 5211
6	8	6. 0000	5. 9188	5. 8376	152. 400	0 23	26. 7645

¹ Standard size of the American National coarse-thread series.

Table 27.—American National 12-pitch thread series

[Pitch, p=0.08333 inch; depth of thread, h=0.05413 inch; basic width of flat, p/8=0.01042 inch; minimum

Identific	ation	В	asic diamete	rs		Thread data	
Sizes	Threads per inch	Major diameter,	Pitch diameter, E	Minor diameter, K	Metric equivalent of major diameter	Helix angle at basic pitch diameter,	Basic area of section at root of thread, $\frac{\pi K^2}{4}$
Inches 1/2 9/16 1 5/8 11/16 3/4	12 12 12 12 12	Inches 0. 5000 0. 5625 0. 6250 0. 6875 0. 7500	Inches 0. 4459 . 5084 . 5709 . 6334 . 6959	Inches 0. 3917 . 4542 . 5167 . 5792 . 6417	mm 12. 700 14. 288 15. 875 17. 463 19. 050	$egin{array}{cccccccccccccccccccccccccccccccccccc$	Square inches 0. 1205 . 1620 . 2097 . 2635 . 3234
13/ ₁₆	12 12 12 12 12	. 8125 . 8750 . 9375 1. 0000 1. 0625	. 7584 . 8209 . 8834 . 9459 1. 0084	. 7042 . 7667 . 8292 . 8917 . 9542	20. 638 22. 225 23. 813 25. 400 26. 988	$\begin{array}{cccc} 2 & 0 \\ 1 & 51 \\ 1 & 43 \\ 1 & 36 \\ 1 & 30 \end{array}$. 3895 . 4617 . 5400 . 6245 . 7151
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	12 12 12 12 12	1. 1250 1. 1875 1. 2500 1, 3125 1. 3750	1. 0709 1. 1334 1. 1959 1. 2584 1. 3209	1. 0167 1. 0792 1. 1417 1. 2042 1. 2667	28. 575 30. 163 31. 750 33. 338 34. 925	1 25 1 20 1 16 1 12 1 9	. 8118 . 9147 1. 0237 1. 1389 1. 2602
$1\frac{7}{16}$ $1\frac{1}{2}$ 2 $1\frac{5}{8}$ $1\frac{3}{4}$ $1\frac{1}{8}$	12 12 12 12 12 12	1. 4375 1. 5000 1. 6250 1. 7500 1. 8750	1. 3834 1. 4459 1. 5709 1. 6959 1. 8209	1. 3292 1. 3917 1. 5167 1. 6417 1. 7667	36. 513 38. 100 41. 275 44. 450 47. 625	$\begin{array}{ccc} 1 & 6 \\ 1 & 3 \\ 0 & 58 \\ 0 & 54 \\ 0 & 50 \end{array}$	1. 3876 1. 5212 1. 8067 2. 1168 2. 4514
$2 \\ 2^{1/8} \\ 2^{1/4} \\ 2^{1/4} \\ 2^{3/8} \\ 2^{1/2} \\ 2^{1/2} $	12 12 12 12 12	2. 0000 2. 1250 2. 2500 2. 3750 2. 5000	1. 9459 2. 0709 2. 1959 2. 3209 2. 4459	1. 8917 2. 0167 2. 1417 2. 2667 2. 3917	50. 800 53. 975 57. 150 60. 325 63. 500	$egin{array}{cccc} 0 & 47 \\ 0 & 44 \\ 0 & 42 \\ 0 & 39 \\ 0 & 37 \\ \end{array}$	2. 8106 3. 1943 3. 6025 4. 0353 4. 4927
2 ⁵ / ₈	12 12 12 12 12	2. 6250 2. 7500 2. 8750 3. 0000 3. 1250	2. 5709 2. 6959 2. 8209 2. 9459 3. 0709	2. 5167 2. 6417 2. 7667 2. 8917 3. 0167	66. 675 69. 850 73. 025 76. 200 79. 375	$\begin{array}{ccc} 0 & 35 \\ 0 & 34 \\ 0 & 32 \\ 0 & 31 \\ 0 & 30 \end{array}$	4. 9745 5. 4810 6. 0119 6. 5674 7. 1475
3½ 3½ 3½ 3½ 3½	12 12 12 12 12	3. 2500 3. 3750 3. 5000 3. 6250 3. 7500	3. 1959 3. 3209 3. 4459 3. 5709 3. 6959	3. 1417 3. 2667 3. 3917 3. 5167 3. 6417	82. 550 85. 725 88. 900 92. 075 95. 250	$\begin{array}{ccc} 0 & 29 \\ 0 & 27 \\ 0 & 26 \\ 0 & 26 \\ 0 & 25 \end{array}$	7. 7521 8. 3812 9. 0349 9. 7132 10. 4159
37/8	12 12 12 12 12	3. 8750 4. 0000 4. 2500 4. 5000 4. 7500	3. 8209 3. 9459 4. 1959 4. 4459 4. 6959	3. 7667 3. 8917 4. 1417 4. 3917 4. 6417	98. 425 101. 600 107. 950 114. 300 120. 650	$\begin{array}{ccc} 0 & 24 \\ 0 & 23 \\ 0 & 22 \\ 0 & 21 \\ 0 & 19 \end{array}$	11. 1433 11. 8951 13. 4725 15. 1480 16. 9217
5 5½ 5½ 6	12 12 12 12 12	5. 0000 5. 2500 5. 5000 5. 7500 6. 0000	4. 9459 5. 1959 5. 4459 5. 6959 5. 9459	4. 8917 5. 1417 5. 3917 5. 6417 5. 8917	127. 000 133. 350 139. 700 146. 050 152. 400	0 18 0 18 0 17 0 16 0 15	18. 7936 20. 7636 22. 8319 24. 9983 27. 2628

Standard size of the American National coarse-thread series.
 Standard size of the American National fine-thread series.

Table 28.—American National 16-pitch thread series

[Pitch, p=0.06250 inch; depth of thread, h=0.04059 inch; basic width of flat, p/8=0.00781 inch; minimum width of flat at major diameter of nut, p/24=0.00260 inch]

	WIC	Ith of hat at	major diame	ter or nut, p	/24=0.00260 in	ienj	
Identific	ation	F	Basic diamete	rs		Thread data	
Sizes	Threads per inch	Major diameter, D	Pitch diameter,	Minor diameter,	Metric equivalent of major diameter	Helix angle at basic pitch diameter,	Basic area of section at root of thread, $\frac{\pi K^2}{4}$
Inches 3/4 1 3/16 7/8 5/16 1	16 16 16 16 16	Inches 0. 7500 . 8125 . 8750 . 9375 1. 0000	Inches 0. 7094 . 7719 . 8344 . 8969 . 9594	Inches 0. 6688 . 7313 . 7938 . 8563 . 9188	mm 19. 050 20. 638 22. 225 23. 813 25. 400	deg min 1 36 1 29 1 22 1 16 1 11	Square inches 0. 3513 . 4200 . 4949 . 5759 . 6630
1½6	16 16 16 16 16	1. 0625 1. 1250 1. 1875 1. 2500 1. 3125	1. 0219 1. 0844 1. 1469 1. 2094 1. 2719	. 9813 1. 0438 1. 1063 1. 1688 1. 2313	26. 988 28. 575 30. 163 31. 750 33. 338	$\begin{array}{cccc} 1 & & 7 \\ 1 & & 3 \\ 1 & & 0 \\ 0 & & 57 \\ 0 & & 54 \end{array}$. 7563 . 8557 . 9612 1. 0729 1. 1907
13/8	16 16 16 16 16	1. 3750 1. 4375 1. 5000 1. 5625 1. 6250	1. 3344 1. 3969 1. 4594 1. 5219 1. 5844	1. 2938 1. 3563 1. 4188 1. 4183 1. 5438	34. 925 36. 513 38. 100 39. 688 41. 275	0 51 0 49 0 47 0 45 0 43	1. 3147 1. 4448 1. 5810 1. 7234 1. 8719
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	16 16 16 16 16	1. 6875 1. 7500 1. 8125 1. 8750 1. 9375	1. 6469 1. 7094 1. 7719 1. 8344 1. 8969	1. 6063 1. 6688 1. 7313 1. 7938 1. 8563	42. 863 44. 450 46. 038 47. 625 49. 213	$\begin{array}{ccc} 0 & 42 \\ 0 & 40 \\ 0 & 39 \\ 0 & 37 \\ 0 & 36 \end{array}$	2. 0265 2. 1873 2. 3542 2. 5272 2. 7064
2 $2\frac{1}{16}$ $2\frac{1}{8}$ $2\frac{3}{16}$ $2\frac{1}{4}$	16 16 16 16 16	2. 0000 2. 0625 2. 1250 2. 1875 2. 2500	1. 9594 2. 0219 2. 0844 2. 1469 2. 2094	1. 9188 1. 9813 2. 0438 2. 1063 2. 1688	50. 800 52. 388 53. 975 55. 563 57. 150	$\begin{array}{ccc} 0 & 35 \\ 0 & 34 \\ 0 & 33 \\ 0 & 32 \\ 0 & 31 \end{array}$	2. 8917 3. 0831 3. 2807 3. 4844 3. 6943
$2\frac{1}{6}$	16 16 16 16	2. 3125 2. 3750 2. 4375 2. 5000	2. 2719 2. 3344 2. 3969 2. 4594	2. 2313 2. 2938 2. 3563 2. 4188	58. 738 60. 325 61. 913 63. 500	$egin{array}{ccc} 0 & 30 \\ 0 & 29 \\ 0 & 29 \\ 0 & 28 \\ \end{array}$	3. 9103 4. 1324 4. 3606 4. 5950
25%	16 16 16 16	2. 6250 2. 7500 2. 8750 3. 0000	2. 5844 2. 7094 2. 8344 2. 9594	2. 5438 2. 6688 2. 7938 2. 9188	66. 675 69. 850 73. 025 76. 200	$egin{array}{ccc} 0 & 26 \\ 0 & 25 \\ 0 & 24 \\ 0 & 23 \\ \end{array}$	5. 0822 5. 5940 6. 1303 6. 6911
3½	16 16 16 16	3. 1250 3. 2500 3. 3750 3. 5000	3. 0844 3. 2094 3. 3344 3. 4594	3. 0438 3. 1688 3. 2938 3. 4188	79. 375 82. 550 85. 725 88. 900	$egin{array}{ccc} 0 & 22 \\ 0 & 21 \\ 0 & 21 \\ 0 & 20 \\ \end{array}$	7. 2765 7. 8864 8. 5209 9. 1799
35/8	16 16 16 16	3. 6250 3. 7500 3. 8750 4. 0000	3. 5844 3. 7094 3. 8344 3. 9594	3. 5438 3. 6688 3. 7938 3. 9188	92. 075 95. 250 98. 425 101. 600	0 19 0 18 0 18 0 17	9. 8634 10. 5715 11. 3042 12. 0614

¹ Standard size of the American National fine-thread series.

Table 29.—Limiting dimensions and tolerances, classes 2 and 3 fits, American National 8-pitch thread series

						Siz	Size (inches)							
Dimensions and tolerances	-13	11/8	11/4	13/8	11/2	15/8	134	17/8	62	21/8	21,4	21/2	61	234
BOLTS AND SCREWS	Įm ab	Inohas	Inohos	Tachoo	Inohoo	Inohoo	Inches	Tachan	46 25	Twohoo	1	1	1	
Classes 2 and 3, major diameter- $\left\{ \begin{array}{ll} \mathrm{Max} \\ \mathrm{Min} \end{array} \right.$	$ \begin{array}{c} 1.0000 & 1.1250 & 1.2500 & 1.3750 & 1.3750 & 1.0000 & 1.0000 & 1.0200 & 1.0200 & 1.0000 \\ 0.0948 & 1.098 & 1.2348 & 1.3598 & 1.4848 & 1.098 & 1.2488 & 1.0098 & 1.0000 & 1.0000 & 1.0000 & 1.0000 \\ 0.0948 & 1.0098 & 1.0000 & 1.0000 & 1.0000 & 1.0000 & 1.0000 & 1.0000 & 1.0000 \\ 0.0152 & 0.0152 & 0.0152 & 0.0152 & 0.0152 & 0.0152 & 0.0152 & 0.0152 & 0.0152 & 0.0152 & 0.0152 \\ \end{array} $. 1250 1098 1. 0152	2500 1 2348 1 0152 0	35981 35981 01520	4848 1 0152 0	6250 6098 0152	1. 7500 1. 7348 1. 0152 (1, 8750 1, 8598 1, 0152	2. 0000 1. 9848 0. 0152	2. 125(2. 109(0. 015)	2. 250 8.2. 234 2. 0. 0154	0 2. 500 8 2. 48 2 0. 01	20.0 18.2.7 52.0.0	348 152 152
Classes 2 and 3, minor diameter. Max.³	•	$8466 \ \ .\ \ 9716 \ \ 1.\ \ 0966 \ \ 1.\ \ 2216 \ \ 1.\ \ 3466 \ \ 1.\ \ 4716 \ \ 1.\ \ 5966 \ \ 1.\ \ 7216 \ \ 1.$. 0966	. 2216	. 3466 1	4716	. 5966	L. 7216	1.8466	8466 1. 9716 2.	3 2. 0966 2.		3466 2. 5	5966
Class 2, pitch diameter (for gen- eral use)Tol Tol	9188 1 . 9112 1 . 9076	$\begin{array}{c} 91881.\ 0438\ 1.\ 1688\ 1.\ 2938\ 1.\ 4188\ 1.\ 5438\ 1.\ 6688\ 1.\ 7938\ 1.\ 9188\ 2.\ 0438\ 2.\ 1688\ 2.\ 4088\ 2.\\ 91121.\ 0359\ 1.\ 1605\ 1.\ 2852\ 1.\ 4098\ 1.\ 5345\ 1.\ 6591\ 1.\ 7838\ 1.\ 9084\ 2.\ 0331\ 2.\ 1578\ 2.\ 4071\ 2.\\ 0076\ 0,\ 0079\ 0,\ 0083\ 0,\ 0086\ 0,\ 0099\ 0,\ 0097\ 0,\ 0100\ 0,\ 0100\ 0,\ 0100\ 0,\ 0117\ 0.\\ \end{array}$	1688 1 1605 1 0083 0	2938 1 2852 1 0086 0	41881. 40981. 00900	5438 5345 0093	L. 6688 1 L. 6591 1 J. 0097 (L. 7938 L. 7838 D. 0100	1. 9188 1. 9084 0. 0104	2. 043 2. 033 0. 010	3 2, 168 1 2, 157 7 0, 011	\$ 2. 418 \$ 2. 407 0. 011	38 2. 6 71 2. 6 17 0. 0	6688 6564 0124
Class 3, pitch diameter	. 9188 . 9134 . 0054	$\begin{array}{c} 91881.\ 0438\ 1.\ 16881.\ 29381.\ 41881.\ 54381.\ 66201.\ 78681.\ 91152.\ 03632.\ 161112.\ 41062.\\ 00540.\ 00556.\ 005580.\ 00610.\ 00656.\ 0.\ 006580.\ 00770.\ 007730.\ 007750.\ 007770.\ 00820.\\ \end{array}$	1688 1 1630 1 0058 0	. 2938 1 . 2877 1 . 0061 0	. 4188 . 4125 . 0063	5438 5373 0065	L. 6688 L. 6620 J. 0068 (L. 7938 L. 7868 D. 0070	1. 9188 1. 9115 0. 0073	2. 0438 2. 0368 0. 0078	\$ 2. 168 \$ 2. 161 5 0. 007	8 2, 418 1 2, 410 7 0, 008	38 30.0 30.0 30.0	6688 6601 0087
NUTS AND TAPPED HOLES												,		
Classes 2 and 3, major diameter. Min.4	$\mathrm{Min.^4\}\ 1.\ 0000\ 1.\ 1250\ 1.\ 2500\ 1.\ 3750\ 1.\ 5000\ 1.\ 6250\ 1.\ 7500\ 1.\ 8750\ 2.\ 0000\ 2.$	1250 1	. 2500	. 3750 1	. 5000 1	6250	l. 7500	L. 8750	2. 0000	2, 1250 2.	2. 2500 2.)2. 500	5000 2. 7	7500
Classes 2 and 3, minor diameter. Max. ⁵	0.8647 8795 1	$8647 \ 0. \ 9897 \ 1. \ 11471. \ 2397 \ 1. \ 36471. \ 48971. \ 61471. \ 73971. \ 86471. \ 98972. \\ 87951. \ 00451. \ 12951. \ 25451. \ 37951. \ 50451. \ 62951. \ 75451. \ 87952. \ 00452. \\ 01480. \ 0$	1147 1295 0148	. 2397 1 . 2545 1 . 0148 0	. 3647 . 3795 . 0148	4897 5045 0148	L. 6147 L. 6295 J. 0148 (L. 7397 L. 7545 J. 0148	1. 8647 1. 8795 0. 0148	1. 989° 2. 0048 0. 0148	7 2. 114 5 2. 129 8 0. 014	. 1147 2. 3647 2. (. 1295 2. 3795 2. (. 0148 0. 0148 0. (17 2. 6 35 2. 6 18 0. 0	$6147 \\ 6295 \\ 0148$
Classes 2 and 3, pitch diameter. Min	. 9188	$9188 \; 1. \; 0438 \; 1. \; 1688 \; 1. \; 2938 \; 1. \; 4188 \; 1. \; 5438 \; 1. \; 6688 \; 1. \; 7938 \; 1. \; 9188 \; 2.$. 1688	. 2938 1	. 4188	5438	. 6688	L. 7938	1. 9188	2. 0438 2.	3 2. 1688 2.		4188 2. 6	8899
Class 2, pitch diameter (for gen- {Max. ⁶ eral use)	. 9264	$92641.\ \ 05171.\ \ 17711.\ \ 30241.\ \ 42781.\ \ 55311.\ \ 67851.\ \ 80381.\ \ 92922.\ \ 05452.\ \ 17982.\ \ 43052.$	$\begin{vmatrix} 1771 \\ 0083 \\ 0 \end{vmatrix}$	3024 1 0086 0	$\begin{array}{c c} 4278 & 1 \\ 0090 & 0 \end{array}$	5531	. 6785 J	l. 8038). 0100	$\frac{1}{0}$, $\frac{9292}{0}$	2. 0548 0. 0107	5 2. 1798 7 0. 0110	32, 430		$6812 \\ 0124$
Class 3, pitch diameter{Tol	• •	$\begin{array}{cccccccccccccccccccccccccccccccccccc$. 1746 1	. 0061 0	. 0063 0	5503	l. 6756 1 . 0068 0	. 8008 . 0070	1. 9261 0. 0073	2. 0518 0. 0078	32. 176	2. 427	70 2. 6 32 0. 0	6775 0087
									-					

			-			Siz	Size (inches)						
Dimensions and folerances	3	31/4	3½	33/4	4	41/4	41/2	434	10	514	545	534	9
BOLTS AND SCREWS	0000	0000	0000	7500	0000	000	000	7500	0000	0020	2000	7500	0000
Classes 2 and 3, major diameter. Min. Tol	2. 9848 3. 0. 0152 0.	2348 3. 0152 0.	4848 3. 0152 0.	73483.01520.0	9848 4 0152 0	2348	L. 4848 0. 0152	4. 7348 0. 0152	2300 3, 5000 3, 7500 4, 8000 4, 2500 4, 7500 3, 7500 3, 7500 3, 2500 3	$\frac{2}{5}$, $\frac{2348}{2348}$	5. 4848 0. 0152	4848 5. 7348 5. 0152 0. 0152 0.	5. 9848 5. 0152
Classes 2 and 3, minor diameter. Max. ³ 2.	8466 3.	0966 3.	3466 3.	5966 3.		8466 4. 0966 4.	L. 3466 4.	4. 5966 4.	4. 8466 5.	5. 0966 5.	5.3466 5.	5. 5966 5.	5.8466
Class 2, pitch diameter (for gen- Max-eral use) Tol.	2. 9188 3. 2. 9058 3. 0. 0130 0.	1688 3. 1556 3. 0132 0.	4188 3. 4055 3. 0133 0.	6688 6554 0134 0	9188 9053 0135	1688 1551 0137	L. 4188 L. 4050 J. 0138	4. 6688 4. 6549 0. 0139	4. 9188 4. 9048 0. 0140	5. 1688 5. 1547 0. 0141	$\begin{array}{c} 1688 \ 3. \ 4188 \ 3. \ 6688 \ 2. \ 9188 \ 4. \ 4188 \ 4. \ 6688 \ 4. \ 9188 \ 5. \ 1688 \ 5. \ 4188 \ 5. \ 6545 \ 5. \ 6545 \ 5. \ 6545 \ 5. \ 6545 \ 5. \ 6545 \ 6. \ 6132 \ 6. \ 6132 \ 6. \ 6137 \ 6137 \ $	5. 6688 5. 6545 0. 0143	5. 9188 5. 9044 0. 0144
Class 3, pitch diameter	2. 9188 3 2. 9096 3 0. 0092 0.	1688 3. 1595 3. 0093 0.	4188 3. 4095 3. 0093 0.	6688 3. 6594 3. 0094 0.	9188 9093 0095	1688 1592 . 0096	L. 4188 L. 4091 J. 0097	4. 6688 4. 6590 0. 0098	4. 9188 4. 9089 0. 0099	5. 1688 5. 1589 0. 0099	$\begin{array}{c} 1688 \ 3. \ 4188 \ 3. \ 6688 \ 3. \ 9188 \ 4. \ 4188 \ 4. \ 6688 \ 4. \ 9188 \ 5. \ 1688 \ 5. \ 4188 \ 5. \ 6688 \ 5. \ 6587 \ 5. \\ 1595 \ 3. \ 6594 \ 3. \ 9093 \ 4. \ 1592 \ 4. \ 4091 \ 4. \ 6590 \ 4. \ 9089 \ 5. \ 1589 \ 5. \ 4088 \ 5. \ 6587 \ 5. \\ 0093 \ 0. \ 0093 \ 0. \ 0095 \ 0. \ 0099 \ 0. \$	5. 6688 5. 6587 0. 0101	5. 9188 5. 9086 5. 0102
NUTS AND TAPPED HOLES													
Classes 2 and 3, major diameter. Min.4	$\mathbf{Min.}^{4} - \frac{1}{3}.\ 0000\ 3.\ 2500\ 3.\ 5000\ 3.\ 7500\ 4.\ 0000\ 4.\ 2500\ 4.\ 5000\ 4.\ 7500\ 5.\ 0000\ 5.\ 2500\ 5.$	2500 3.	5000 3.	7500 4.	0000	. 2500	L. 5000	4. 7500	5. 0000	5.2500	5. 5000	5000 5. 7500 6.	6. 0000
Classes 2 and 3, minor diameter. $\left\{ \begin{array}{l} \mathrm{Min} \\ \mathrm{Tol} \end{array} \right\}$	0.00	1147 3. 1295 3. 0148 0.	$\begin{array}{c} 3647 \\ 3795 \\ 3.0148 \\ 0.0148 \\ 0.0148 \\ 0.0000 \\ 0.00000 \\ 0.000000 \\ 0.00000000$	6147 6295 0148 0	8647 4 8795 4 0148 0	. 1147 . 1295 . 0148	L. 3647 L. 3795 J. 0148	4. 6147 4. 6295 0. 0148	4. 8647 4. 8795 0. 0148	5. 1147 5. 1295 0. 0148	$\begin{array}{c} 8647 \ 3. \ 1147 \ 3. \ 3647 \ 3. \ 6147 \ 3. \ 8647 \ 4. \ 1147 \ 4. \ 6147 \ 4. \ 8647 \ 5. \ 1147 \ 5. \ 8647 \ 5. \ 6147 \ 5. \ 6147 \ 5. \ 6147 \ 6. \ 6147 \ 6. \ 6148 \ 6148 \ 6$	5. 6147 5. 6295 0. 0148	5. 8647 5. 8795 5. 0148
Classes 2 and 3, pitch diameter. Min.	2. 9188 3.	1688 3.	4188 3.	6688 3.	9188 4.	. 1688 4.	L. 4188 4.	4. 6688 4.		9188 5, 1688 5.	5. 4188 5.	5. 6688 5.	5. 9188
Class 2, pitch diameter (for gen- {Max. ⁶ -eral use)	0.0	$1820 \ 3.0132 \ 0.0132 \ 0.0000$	$\begin{array}{c c} 4321 & 3. \\ 0133 & 0. \end{array}$	6822 3.000	$\frac{9323}{0135}$. 1825	L. 4326). 0138	4. 6827 0. 0139	4. 9328 0. 0140	$5.1829 \\ 0.0141$	$9318 \ \ 3.1820 \ \ 3.4321 \ \ 3.6822 \ \ 3.9323 \ \ 4.1825 \ \ 4.4326 \ \ 4.6827 \ \ 4.9328 \ \ 5.1829 \ \ 5.4330 \ \ 5.6831 \ \ 5.0139 \ \ 0.0132 \ \ 0.0132 \ \ 0.0134 \ \ 0.0135 \ \ 0.0137 \ \ 0.0137 \ \ 0.0137 \ \ 0.0139 \ \ 0.0140 \ \ 0.0141 \ \ 0.01420 \ \ 0.0143 \ \$	5. 6831 0. 0143	5. 9332 5. 0144
Class 3, pitch diameter $\left\{\frac{\mathrm{Max.^6_{}}}{\mathrm{Tol_{}}}\right\}$ 0.	3 2. 9280 3. 0. 0092 0.	1781 3. 0093 0.	4281 3. 0093 0.	6782 3. 0094 0.	9283 4 0095 0	. 1784	L. 4285	4. 6786 0. 0098	4. 9287 0. 0099	5. 1787 0. 0099	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$\frac{5.6789}{0.0101}$	5. 9290 5. 0102
					:			1777	,				0,000

1 Pitch diameter tolerances include errors of lead and angle. The class 2 tolerances are based on the formulas in table 116 and a length of engagement equal to the basic major diameter for sizes from 1/8 to 3 inches, inclusive, and a length of engagement of 3 inches for sizes over the 3-inch. The class 3 tolerances are 70 percent of the class 2 tolerances. The 1-inch size being in the American National coarse-thread series, the tolerances for this size correspond to that series.

¹ Standard size screw and nut of the American National coarse-thread series.

¹ Dimensions given for the maximum minor diameter of the serew are figured to the nicersection of the worn tool are with a center line through creek and rook. The minimum minor diameter of the serew shall be that corresponding to a flat at the minor diameter.

of the minimum screw equal to $\frac{1}{2}$ Xp, and may be determined by subtracting 0.0812 inch from the minimum pitch diameter of the screw † Dimensions for the minimum major diameter of the nut correspond to the basic flat

4 Dimensions for the minimum major diameter of the nut correspond to the basic flat (\$4.\$\tilde{x}\$), and the profile at the major diameter produced by a worn tool must not fall below the basic outline. The maximum major diameter of the nut shall be that corresponding to a flat at the major diameter of the maximum nut equal to \(\frac{1}{2}\tilde{x}\), and may be ditermined by adding 0.0499 inch to the maximum pitch diameter of the nut.

⁵ Present Army ordnance practice follows Handbook H25 and the mimeographed Supplement to Handbook H28 in the maximum minor diameters of nuts.

⁶ These dimensions are the minimum metal or "not go" size. The "go" or basic size the one that should be placed on the component drawing with the tolerance.

Table 30.—Limiting dimensions and tolerances, classes 2 and 3 fits, American National 12-pitch thread series

							Size	te (inches)							
Dimensions and tolerances ¹	7,5	9/16 2	28%	11/16	3%	13/16	8%	15/16	1	11/16	11/83	13/16	1143	15/16	13/8 3
Bolts and Screws Classes 2 and 3, Min- major diam [Tol	EWS Inches Inches	Inch 0. 5625 . 5513 . 0112	Inch 0. 6250 6138 . 0112	Inch 0. 6875 0. 6763 0. 0112	Inch 0. 7500 7388 . 0112	Inch 3. 8125 8013 . 0112	Inch 3. 8750 (8638 . 0112 (Inch 0. 9375 0. 9263 0. 0112	Inch 1. 0000 3. 9888 . 0112	Inches 1. 0525 1. 0513 0. 0112	Inches 1. 1250 1. 1138 0. 0112	Inches 1. 1875 1. 1763 0. 0112	Inches 1. 2500 1. 2388 0. 0112	Inches 1. 3125 1. 3013 0. 0112	Inches 1. 3750 1. 3638 0. 0112
Classes 2 and 3, minor diam.	. 3978	. 4603	. 5228	. 5853	. 6478	. 7103	. 7728	. 8353	. 8978	. 9603	1. 0228	1. 0853	9603 1. 0228 1. 0853 1. 1478 1.	1. 2103 1.	1. 2728
Class 2, pitch diam. $\left\{\begin{array}{l} Max-\\ Min-\\ Tol \end{array}\right\}$. 4459 . 4403 . 0056	. 5084 . 5028 . 0056	. 5709 . 5653 . 0056	. 6334 . 6278 . 0056	. 6959 . 6903 . 0056	. 7584 . 7528 . 0056	$\begin{array}{c} 8209 \\ 8153 \\ 0056 \end{array}$. 8834 . 8778 . 0056		1. 0084 1. 0028 0. 0056	$\begin{array}{c} .9459\ 1.\ 0084\ 1.\ 0709\ 1. \\ .9403\ 1.\ 0028\ 1.\ 0653\ 1. \\ .0056\ 0.\ 0056\ 0.\ 0056\ 0. \end{array}$	1. 1334 1. 1. 1278 1. 0. 0056 0.	1. 1959 1. 1. 1903 1. 0. 0056 0.	1. 2584 1. 3. 2528 1. 3. 0. 0056 0. 0	1. 3209 1. 3153 0. 0056
Class 3, pitch diam. Min.	. 4459 . 4419 . 0040	. 5084 . 5044 . 0040	. 5709 . 5669 . 0040	. 6334 . 6294 . 0040	. 6959 . 6919 . 0040	. 7584 . 7544 . 0040	. 8209 . 8169 . 0040	. 8834 . 8794 . 0040	. 9459 . 9419 . 0040	1. 0084 1. 0044 0. 0040	. 9459 1. 0084 1. 0709 1. . 9419 1. 0044 1. 0669 1. . 0040 0. 0040 0. 0040 0.	1. 1334 1. 1294 0. 0040	. 1334 1. 1959 1. 2 . 1294 1. 1919 1. 2 . 0040 0. 0040 0. 0	1. 2584 1. 3 1. 2544 1. 3 0. 0040 0. 0	1. 3209 1. 3169 0. 0040
Nurs and Tapped Holes Classes 2 and 3, major diamMin. ⁵	. 5000	. 5625	. 6250	. 6875	. 7500	. 8125	. 8750	. 9375	9375 1. 0000 1. 0625 1.	1. 0625	1. 1250	1250 1. 1875 1.	1. 2500 1	1. 3125 1	1. 3750
Classes 2 and 3, Min minor diam Tol	. 4098 . 4225 . 0127	$\begin{array}{c} 4723 \\ 4850 \\ 0127 \end{array}$. 5348 . 5438 . 0090	. 5973 . 6063 . 0090	. 6598 . 6688 . 0090	. 7223 . 7313 . 0090	. 7848 . 7938 . 0090	. 8473 . 8563 . 0090	3. 9098 . 9188 . 0090	0. 9723 . 9813 . 0090	1. 0348 1. 0438 0. 0090	1. 0973 1. 1063 0. 0090	$\begin{array}{c} 8473 \ 0.\ 9098 \ 0.\ 9723 \ 1.\ 0348 \ 1.\ 0973 \ 1.\ 1598 \ 1.\ 2223 \ 1. \\ 8563 \ .\ 9188 \ .\ 9813 \ 1.\ 0438 \ 1.\ 1063 \ 1.\ 1688 \ 1.\ 2313 \ 1. \\ 0090 \ .\ 0090 \ 0.\ 0090 \ 0.\ 0090 \ 0.\ 0090 \ 0.\ 0090 \ 0.\ 0090 \ 0. \end{array}$	1. 2223 1. 2313 0. 0090	1. 2848 1. 2938 0. 0090
Classes 2 and 3, pitch diamMin	. 4459	. 5084	. 5709	. 6334	. 6959	. 7584	. 8209	. 8834	. 9459	1. 0084	1. 0709	1. 1334	. 9459 1. 0084 1. 0709 1. 1334 1. 1959 1. 2584 1. 3209	1. 2584	1. 3209
Class 2, pitch diam. [Max. ⁶ - (for general use)_[Tol	. 4515	$\frac{5140}{0056}$. 5765	. 6390	. 7015	. 7640	. 8265	. 8890		1. 0140 0. 0056	1. 0765 0. 0056	1. 1390 0. 0056	$\begin{array}{c} .\ 9515\ 1.\ 0140\ 1.\ 0765\ 1.\ 1390\ 1.\ 2015\ 1.\ 2640\ 1.\ 3265 \\ .\ 0056\ 0.\ 0056\ 0.\ 0056\ 0.\ 0056\ 0.\ 0056 \end{array}$	$\frac{1.2640}{0.0056}$	1. 3265 3. 0056
Class 3, pitch diam_{Tol 0040	. 4499		$\begin{array}{c c} .5124 & .5749 \\ .0040 & .0040 \\ \end{array}$. 6374	. 6374 . 6999 . 7624 . 0040 . 0040 . 0040	. 7624	. 8249	. 8874	9499	1. 0124 0. 0040	1. 0749 0. 0040	1. 1374 0. 0040	$\begin{array}{c} .\ 9499 \ 1.\ 0124 \ 1.\ 0749 \ 1.\ 1374 \ 1.\ 1999 \ 1.\ 2624 \ 1. \\ .\ 0040 \ 0.\ 0040 \ 0.\ 0040 \ 0.\ 0040 \ 0.\ 0040 \ 0.\ 0040 \ 0. \end{array}$	1. 2624	1. 3249 0. 0040

Difficultions and potentialess -		-														THE REAL PROPERTY AND ADDRESS OF THE PERSON.	
	17/16	11/2 3	15/8	134	178	73	21/8	21,4	23/8		21/2	25%	23/4	27/8	eo	66	378
DOLL'S AND SCREWS	2	Tookas	o year	Tachoo	Top to	Tookee	Tookoo	Too P	10.07	1 2		100	1	,			
Classes 2 and 3, $\begin{cases} Max - 1.7 \\ Min - 1.1 \end{cases}$ major diam $\begin{cases} Tol_{} & 0.7 \\ Tol_{} & 0.7 \end{cases}$	4375 1. 4263 1. 0112 0.	$\begin{array}{c} 1.4375 \\ 1.4263 \\ 1.6 \\ 0.0112 \\$	6250 1. 6138 1. 0112 0.	7500 7388 0112	1. 8750 1. 8638 1. 0112	2. 0000 1. 9888 0. 0112	2. 1250 2. 1138 0. 0112	2. 250 2. 238 0. 011	0 2. 37 8 2. 36 2 0. 01	1202. 1202. 120.	1112 0.	6250 6138 0112	2. 7500 2. 7388 0. 0112	2. 875 32. 863 0. 011	0 3. 000 8 2. 988 2 0. 011	8 3 1 1 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	250 138 112
Classes 2 and 3, minor diamMax.*_1. 3353 1. 3978 1. 5228 1. 6478 1. 7728 1. 8978 2. 0228 2. 1478 2. 2728 2. 3978 2. 5228 2. 6478 2. 7728 2. 8978 3.	3353 1.	3978 1.	5228 1.	6478 1	L. 7728	1. 8978	2. 0228	2. 147	8 2. 27	28 2. 5	978 2.	5228	2. 6478	3 2. 772	8 2. 897	8.3	0228
Class 2, pitch diam. $\begin{cases} Max_{} & \text{I. 8} \\ Min_{} & \text{I. 9} \end{cases}$ (for general use) - \begin{bmatrix} Tol_{} & 0. \end{bmatrix}	3834 1. 3778 1. 0056 0.	$\begin{array}{c} 3834 \ 1. \ 4459 \ 1. \ 5709 \ 1. \ 6959 \ 1. \ 8209 \ 1. \ 8459 \ 2. \ 0709 \ 2. \ 1959 \ 2. \ 3209 \ 2. \ 4459 \ 2. \ 3778 \ 1. \ 4403 \ 1. \ 5645 \ 1. \ 8645 \ 1. \ 8143 \ 1. \ 9392 \ 2. \ 0641 \ 2. \ 1890 \ 2. \ 3139 \ 2. \ 4388 \ 2. \ 0056 \ 0. \ 0006 \ 0. \ 0006 \ 0. \ 0006 \ 0. \ 0007 \ 0. \ 0071 \ 0. \ 0. \ 0. \ 0. \ 0. \ 0. \ 0. \ 0$	5709 1. 5645 1. 0064 0.	6959 6894 0065	1. 8209 1. 8143). 0066	1. 9459 1. 9392 0. 0067	2. 0709 2. 0641 0. 0068	2. 195 2. 189 0. 006	9 2. 32 0 2. 31 9 0. 00	39 2. 70 0.	1459 2. 1388 2. 1071 0.	. 5709 . 5638 . 0071	. 5709 2. 6959 2. 8 . 5638 2. 6887 2. 8 . 0071 0. 0072 0. 0	2. 820 2. 813 0. 007	8209 2. 9459 3. 8136 2. 9385 3. 0073 0. 0074 0.		0709 0635 0074
Class 3, pitch diam $\left\{ \begin{array}{l} \text{Max}_{-} & 1. \\ \text{Min}_{-} & 1. \\ \text{Tol}_{-} & 0. \end{array} \right\}$	3834 1. 3794 1. 0040 0.	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	5709 1. 5664 1. 0045 0.	6959 6913 0046	1. 8209 1. 8163 3. 0046	1. 9459 1. 9412 0. 0047	2. 0709 2. 0661 0. 0048	2. 195 2. 191 0. 004	92. 32 12. 31 80. 00	809 2. 60 2. 49 0.	1459 2 1410 2. 1049 0.	5709 2. 5659 2. 0050 0.	5709 2. 6959 2. 5659 2. 6909 2. 0050 0. 0050 0.	2. 820 2. 815 0. 005	$\begin{array}{c} 8209 \ 2. \ 945 \\ 8158 \ 2. \ 940 \\ 0051 \ 0. \ 005 \end{array}$	$\begin{array}{c c} 9459 & 3. & 0 \\ 9408 & 3. & 0 \\ 0051 & 0. & 0 \end{array}$	0709 0657 0052
NUTS AND TAPPED HOLES																	
Classes 2 and 3, major diamMin.6- 1.	4375 1.	5000 1. 6250 1. 7500 1. 8750 2. 0000 2.	6250 1.	7500	1. 8750	2. 0000	2. 1250 2.		2500 2. 37	3750 2.	5000 2.	6250 2.	2. 7500 2.		8750 3. 0000 3.		1250
Classes 2 and 3, Min. 1. 1. minor diam. Tol. 1. 701.	3473 1. 3563 1. 0090 0.	$\begin{array}{c} 34731.\ 40981.\ 5348 1.\ 65981.\ 78481.\ 90982.\ 034882.\\ 35631.\ 41881.\ 54381.\ 66881.\ 79381.\ 91882.\ 04382.\\ 00900.\ 00900.\ 00900.\ 00900.\ 00900.\ 00900.\\ \end{array}$	5348 1. 5438 1. 0090 0.	6598 6688 0090	1. 7848 1. 7938 1. 0090	1. 9098 1. 9188 0. 0090	2. 0348 2. 0438 0. 0090	2. 1598 2. 2. 1688 2. 0. 0090 0.		2848 2. 4 2938 2. 4 0090 0. 0	4098 2. 4188 2. 0090 0.	5348 2. 5438 2. 0090 0.	2. 6598 2. 2. 6688 2. 0. 0090 0.	6598 2. 7848 2. 6688 2. 7938 2. 0090 0. 0090 0.	7848 2. 9098 3. 7938 2. 9188 3. 0090 0. 0090 0.		0348 0438 0090
Classes 2 and 3, pitch diamMin 1.	3834 1.	- 1. 3834 1. 4459 1. 5709 1. 6959 1. 8209 1. 9459 2. 0709 2. 1959 2. 3209 2. 4459 2. 5709 2. 6959 2. 8209 2. 9459 3.	5709 1.	6959	. 8209	1. 9459	2. 0709	2. 195	9 2. 32	09 2.	1459 2.	5709	2. 6959	2. 820	9 2. 945	93.0	020
Class 2, pitch diam. $\{Max.^6 1.8\}$ (for general use) $\{Tol_{}, 0.8\}$	3890 1. 0056 0.	$38901.\ 45151.\ 57731.\ 70241.\ 82751.\ 95262.\ 07772.\ 20282.\ 32792.\ 45302.\ 57802.\ 70312.\ 82822.\ 00560.\ 00560.\ 006640.\ 006660.\ 006670.\ 006670.\ 006680.\ 006690.\ 007010.\ 00710.\ 007120.\ 00730.$	5773 1. 0064 0.	7024 0065	1. 8275	1. 9526 0. 0067	2. 0777 0. 0068	2. 202 0. 006	8 2. 32 9 0. 06	79 2. 4	1530 2. 1071 0.	5780	2. 7031 0. 0072	2. 828	2 2. 9533 3. 3 0. 0074 0.		0783 0074
Class 3, pitch diam. { Max. ⁶ -1. 3 (Tol 0.	3874 1.0040 0.0	$\begin{array}{c} 2.6-1.38741.44991.57541.70051.82551.95062.07572.20072.32582.45082.57592.70092.82602.95103.575910.004010.004510.004510.004610.004610.00470.004810.004810.004910.004910.005010.005010.005010.005110.00$	5754 1. 0045 0.	7005 0046 0	. 8255	1. 9506 0. 0047	2. 0757 0. 0048	2. 200	72.32	58 2. 4	1508 2. 1049 0.	5759	2. 700g 0. 0050	2. 826 0. 005	02. 951 10. 005		$0761 \\ 0052$

Table 30.—Limiting dimensions and tolerances, classes 2 and 3 fits, American National 12 pitch-thread series—Continued

	9	nches 0000 9888	0112	$9459 \\ 9371 \\ 0088$	$9459 \\ 9397 \\ 0062$
	534	ches 17500 6.	0112 0.	6959 5. 6872 5. 0087 0.	6959 5. 6898 5. 3061 0.
		hes 0000 5. Im 888 5.	112 0. (978 5. (459 5. 373 5. 086 0.	459 5. 399 5. 060 0.
	51/2	5. 5. 4.	0.0	7.7.0 4.4.0	4.6.0
	5%	Inches 5. 2500 5. 2388). 0112 5. 1478	5. 1959 5. 1874 6. 0085	5. 1959 5. 1900 6. 0059
	2	Inches 00000 9888	. 0112(9459	9459
	43,4	nches 7500 5 7388 4	0112 0	6959 4 6876 4 0083 0	$\begin{array}{c c} 6959 & 4 \\ 6901 & 4 \\ 9058 & 0 \end{array}$
		300 4. 4. 4. 4. 4. 4. 4. 4. 4. 4. 4. 4. 4.	[2 0. 78 4.	59 4. 31 0.	59 4. 57 0.
	41/2	Inche 4. 500 4. 488	0. 01. 4. 397	4. 448 4. 437 0. 008	4. 448 4. 446 0. 008
Size (inches)	41/4	Inches 2500 2388	1478	1959 1879 0080	1959 1903 0056
Size	4	10000 4 9888 4	0112 0 $8978 4$	9459 4 9380 4 0079 0	9459 4. 9404 4. 0055 0.
		300 300 300 400 100 100 100 100 100 100 100 100 1	<u> </u>	8 2 3 3 2 3 3 3	00 00 00 00 00 00 00 00 00 00 00 00 00
	378	Inche. 878.). 01.). 77.	826 826 907	. 82(. 81(. 00(
	33/4	7500 3 7388 3	01112 0	6959 3 6881 3 0078 0	6959 3 6905 3 0054 0
		38037	$\frac{12}{28}$ $\frac{28}{3}$.	09 3. 77 0.	093. 5533. 540.
	358	Inche 3. 62 3. 61	0. 01 3. 52	3. 57 3. 56 0. 00	3. 57 3. 56 0. 00
	31/2	Inches 5000 4888	3978	4459 4383 0076	4459 4406 0053
	338	BOLITS AND SCREWS Inches Inche	minor Max.4 3. 1478 3. 2728 3. 3978 3. 5228 3. 6478 3. 7728 3. 8978 4. 1478 4. 3978 4. 6478 4. 8978 5. 1478 5. 3978 5. 6478 5. 8978	$\begin{array}{c} 3. \ 1959 \ 3. \ 3209 \ 3. \ 3209 \ 3. \ 4459 \ 3. \ 5709 \ 3. \ 6959 \ 3. \ 8209 \ 3. \ 9459 \ 4. \ 4459 \ 4. \ 4459 \ 4. \ 4459 \ 4. \ 9459 \ 6. \ 9459 \ 5. \ 9459 \ 5. \ 9459 \ 5. \ 9459 \ 5. \ 9459 \ 5. \ 9459 \ 6. \ 9459 \ 9459 \ 6. \ 9459 \ 6. \ 9459 \ 6. \ 9459 \ 6. \ 9459 \ 6. \ 94590 \ 94590 \ 94590 \ 94590 \ 94590 \ 94590 \ 94590 \ 94590 \ 94590 \ 94590 \ 94590 \ 94590 $	Class 3, pitch $\{\text{Max}_{}\}$ 3.1959 3.3209 3.4459 3.5709 3.6959 3.8209 3.9459 4.1959 4.4459 4.6959 4.9459 5.1959 5.4459 5.6959 5.9459 diam.
	31/4	Inches 2500 3 2388	0112	1959 1884 0075	1959 1907 0052
		70000	<u>් ස</u> ;	<u> </u>	<u> </u>
Dimensions and tolerances ¹		Max-	Classes 2 and 3, minor diam.	Class 2, pitch Max-diam (for gen- Min-eral usc)	Max- Min Tol
104	of Due s	BOLTS AND SCREWS SSES 2 and 3, Max.	and 3	pitch r gen-	pitch
	HOISIIG	L'rs	asses 2	$\frac{2}{1}$, (for use).	65,
Dimer		Bo: Classes maio	Classes	Class diam eral	Class

may be determined by subtracting

			201	000.110	
	- 3. 2500 3. 3750 3. 5000 3. 6250 3. 7500 3. 8750 4. 0000 4. 2500 4. 5000 4. 7500 5. 0000 5. 2500 5. 5000 5. 7500 6. 0000	$\begin{array}{c} 3.\ 1598\ 3.\ 2848\ 3.\ 4098\ 3.\ 5348\ 3.\ 6598\ 3.\ 7848\ 3.\ 9098\ 4.\ 1598\ 4.\ 4098\ 4.\ 4188\ 4.\ 6688\ 4.\ 9188\ 5.\ 4188\ 5.\ 4188\ 5.\ 4188\ 5.\ 4188\ 4.\ 4188\$	- 3. 1959 3. 3209 3. 4459 3. 5709 3. 6959 3. 9459 4. 1959 4. 4459 4. 6959 4. 9459 5. 1959 5. 4459 5. 6959 5. 9459	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$\begin{array}{c} 3.\ 2011\ 3.\ 3262\ 3.\ 4512\ 3.\ 5763\ 3.\ 7013\ 3.\ 8264\ 3.\ 9514\ 4.\ 2015\ 4.\ 4516\ 4.\ 7017\ 4.\ 9518\ 5.\ 2018\ 5.\ 4519\ 5.\ 7020\ 5.\ 9521 \\ \hline0.\ 0.055\ 0.\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\ 0.\ 0.055\$
	7500	6598 2 6688 7 0090	6929	7046	7020
	5000 5.	4098 5. 4188 5. 0090 0.	4459 5.	4545 5. 0086 0.	4519 5. 0060 0.
	500 5.	598 5. 688 5. 090 0.	959 5.	044 5. 085 0.	018 5. 059 0.
	00 5. 2	98 5. 1 88 5. 1 90 0. 0	59 5. 1	143 5. 2 184 0. 0	18 5. 2
	00 5. 00	88 4. 91 0 0. 00	69 4. 94	24. 95 33 0. 00	7 4. 95 88 0. 00
-	4. 750	34. 659 34. 668 0. 009	4. 695	4. 704	4. 701
	4. 5000	4. 4098 4. 4188 0. 0090	4. 4459	4. 4540 0. 0081	4. 4516 0. 0657
	. 2500	. 1598 . 1688 . 0090	. 1959	2039	2015
	0000	9098 4 9188 4 0090 0	94594	9538 4	9514 4
	3750 4.	7848 3. 7938 3. 0090 0.	3209 3.	3287 3. 0078 0.	3264 3. 0055 0.
	500 3. 8	598 3. 7 388 3. 7 390 0. 0	959 3. 8)37 3. 8)78 0. (013 3. 8
	50 3. 78	48 3. 68 38 3. 60 90 0. 00	09 3. 6	36 3. 70 77 0. 00	63 3. 7. 54 0. 0
	0 3. 62	8 3. 53 8 3. 54 0 0. 00	9 3. 57	5 3. 57	2 3. 57 3 0. 00
	3. 500	3. 409 3. 418 0. 009	3.445	3. 453 0. 007	3. 451 0. 005
	3. 3750	3. 2848 2. 2938 0. 0090	3. 3209	3. 3285 3. 0076	3. 3262). 0053
	. 2500	1598 1688 0090	. 1959	2034	2011
Hores	major Min. ⁵	Min 3 Max 3 Tol	pitch Min3	Max. ⁶ 3 Tol0	ch {Max. ⁶ 3
NUTS AND TAPPED HOLES	Classes 2 and 3, major diamMin. ⁵	Class 2 and 3, Minminor diam Tol	Classes 2 and 3, pitch diamMin	Class 2, pitch $\max_{\text{dian (for gen-eral use)}} Tol$	pit
TS AND	sses 2	ss 2 a	sses 2	ss 2, iam (fo ral use)	ass 3, diam
No	Cla	Cla	Cla	Cla	Cla

mum minor diameter of the screw shall be that corresponding to a flat at the minor diameter of the minimum screw equal to $18 \times p$, and The class 2 tolerances for of 6 threads or 1/2 inch. The class 3 tolcrances are 70 percent of the class 2 tolerances. For lengths of engagement of 1 inch, 0.0010 inch may be added to these tolerances. As certain sizes up to 1½ inches are included in the American National coarse- or fine-thread series, the tolerances to and including 1½ inches correspond to those series. sizes above $1\frac{1}{2}$ inches are based on the formulas in table $1\frac{1}{6}$ and a length of engagement 1 Pitch-diameter tolerances include errors of lead and angle.

4 Dimensions given for the maximum minor diameter of the screw are figured to the tersection of the worn tool are with a center line through crest and root. The mini-Standard-size screw and nut of the American National coarse-thread series. Standard-size screw and nut of the American National fine-thread series. intersection of the worn tool are with a center line through crest and root.

sponding to a flat at the major diameter of the maximum nut equal to $154 \times p_0$, and may be determined by adding 0.062 inch to the maximum pitch diameter of the nut.

§ These dimensions are the minimum metal or "not go" size. The "go" or basic size is the one that should be placed on the component drawing with the tolerance. flat (38Xp) and the profile at the major diameter produced by a worn tool must not fall below the basic outline. The maximum major diameter of the nut shall be that correb Dimensions for the minimum major diameter of the nut correspond to the basic 0.0541 inch from the minimum pitch diameter of the screw

Table 31.—Limiting dimensions and tolerances, classes 2 and 3 fit, American National 16-pitch-thread series

	a de la companya de				Size (inches)	ches)				
Ulmensions and folerances	34.2	13/16	%	15/16	1	11/16	13/8	13/16	11/4	15/16
Bolts and Screws [Max	Inch 0. 7500 . 7410 . 0090	$^{Inch}_{0.8125}$ $^{8035}_{0.000}$	$^{Imch}_{0.\ 8750}_{0.\ 8660}_{0.\ 0090}$	Inch 0. 9375 . 9285 . 0090	$\frac{Imch}{1.0000}$ 0.9910 0.0990	Inches 1. 0625 1. 0535 0. 0090	$ \begin{array}{c} Inches \\ 1.1250 \\ 1.1160 \\ 0.0090 \\ \end{array} $	Inches 1. 1875 1. 1785 0. 0090	Inches 1. 2500 1. 2410 0. 0090	Inches 1. 3125 1. 3035 0. 0090
Minor diameter Max 3	. 6733	. 7358	. 7983	8098	. 9233	. 9858	1.0483	1. 1108	1. 1733	1. 2358
Class 2, pitch diameter (for Maxgeneral use)	. 7094 . 7049 . 0045	. 7719 . 7668 . 0051	. 8344 . 8293 . 0051	. 8969 . 8917 . 0052	. 9594 . 9542 . 0052	1, 0219 1, 0166 0, 0053	1. 0844 1. 0799 0. 0054	1. 1469 1. 1415 0. 0054	1. 2094 1. 2039 0. 0055	1. 2719 1. 2664 0. 0055
Class 3, pitch diameter Min Tol	. 7094 . 7062 . 0032	. 7719 . 7684 . 0035	. 8344 . 8308 . 0036	. 8969 . 8933 . 0036	. 9594 . 9557 . 9037	$\begin{array}{c} 1.\ 0219 \\ 1.\ 0182 \\ 0.\ 0037 \end{array}$	1. 0844 1. 0806 0. 0038	1. 1469 1. 1431 0. 0038	1. 2094 1. 2056 0. 0038	1. 2719 1. 2680 0. 0039
NUTS AND TAPPED HOLES										
Major diameter Min 4	. 7500	. 8125	. 8750	. 9375	1.0000	1.0625	1. 1250	1. 1875	1. 2500	1. 3125
Minor diameter ${\rm Max}^{\rm 6}$. 6823 . 6903 . 0080	. 7448 . 7528 . 0080	. 8073 . 8153 . 6080	. 8698 . 8778 . 0980	0. 9323 . 9403 . 0080	0. 9948 1. 0028 0. 0080	1. 0573 1. 0653 0. 0080	1. 1198 1. 1278 0. 0080	1. 1823 1. 1903 0. 0080	1. 2448 1. 2528 0. 0080
Class 2, pitch diameter (for Min general use)	. 7094 . 7139 . 0045	. 7719 . 7770 . 0051	. 8344 . 8395 . 0051	. 8969 . 9021 . 0052	9594 9646 0052	1. 0219 1. 0272 0. 0053	1. 0844 1. 0898 0. 0054	1. 1469 1. 1523 0. 0054	1. 2094 1. 2149 0. 0055	1. 2719 1. 2774 0. 0055
Class 3, pitch diameter{Max	. 7094 . 7126 . 0032	. 7719 . 7754 . 0035	. 8344 . 8380 . 0036	. 8969 . 9005 . 0036	. 9594 . 9631 . 0037	1. 0219 1. 0256 0. 0037	1. 0844 1. 0882 0. 0038	1. 1469 1. 1507 0. 0038	1, 2094 1, 2132 0 0038	1. 2719 1. 2758 0. 0039

						Size (inches)	ches)				
Difficultions and tolerances .		13%	17/16	11/2	19/16	15%	111/16	134	13/16	178	115/16
Bolts and Screws Major diameter	Max Min Tol	Inches 1. 3750 1. 3660 0. 0090	Inches 1. 4375 1. 4285 0. 0090	Inches 1. 5000 1. 4910 0. 0090	Inches 1. 5625 1. 5535 0. 0090	Inches 1. 6250 1. 6160 0. 0090	Inches 1. 6875 1. 6785 0. 0090	Inches 1. 7500 1. 7410 0. 0090	Inches 1. 8125 1. 8035 0. 0090	Inches 1. 8750 1. 8660 0. 0090	Inches 1. 9375 1. 9285 0. 0090
Minor diameter	Max.3	1, 2983	1. 3608	1, 4233	1. 4858	1. 5483	1. 6108	1. 6733	1. 7358	1. 7983	1.8608
Class 2, pitch diameter (for gen- eral use)	[Max [Min [Tol	1. 3344 1. 3288 0. 0056	1. 3969 1. 3913 0. 0056	1. 4594 1. 4537 0. 0057	1. 5219 1. 5161 0. 0058	1. 5844 1. 5786 0. 0058	1. 6469 1. 6411 0. 0058	1. 7094 1. 7035 0. 0059	1. 7719 1. 7660 0. 0059	1. 8344 1. 8284 0. 0060	1. 8969 1. 8909 0. 0060
Class 3, pitch diameter{	[Max Min	1. 3344 1. 3305 0. 0039	1. 3969 1. 3929 0. 0040	1. 4594 1. 4554 0. 0040	1. 5219 1. 5179 0. 0040	1. 5844 1. 5803 0. 0041	1. 6469 1. 6428 0. 0041	1. 7094 1. 7053 0. 0041	1, 7719 1, 7677 0, 0042	1. 8344 1. 8302 0. 0042	1. 8969 1. 8927 0. 0042
NUTS AND TAPPED HOLES	ŭ										
Major diameter	Min.4	1. 3750	1, 4375	1. 5000	1, 5625	1. 6250	1. 6875	1, 7500	1.8125	1,8750	1. 9375
Minor diameter{	Min Max. ⁵ Tol	1. 3073 1. 3153 0. 0080	1. 3698 1. 3778 0. 0080	1. 4323 1. 4403 0. 0080	1. 4948 1. 5028 0. 0080	1. 5573 1. 5653 0. 0080	1. 6198 1. 6278 0. 0080	1. 6823 1. 6903 0. 0080	1. 7448 1. 7528 0. 0080	1. 8073 1. 8150 0. 0080	1. 8698 1. 8778 0. 0080
Class 2, pitch diameter (for general use)	$\begin{cases} \text{Min}_{} \\ \text{Max}_{} \\ \text{Tol}_{} \end{cases}$	1. 3344 1. 3400 0. 0056	1. 3969 1. 4025 0. 0056	1. 4594 1. 4651 0. 0057	1. 5219 1. 5277 0. 0058	1. 5844 1. 5902 0. 0058	1. 6469 1. 6527 0. 0058	1. 7094 1. 7153 0. 0059	1. 7719 1. 7778 0. 0059	1. 8344 1. 8404 0. 0060	1. 8969 1. 9029 0. 0060
Class 3, pitch diameter	Min Max Tol	1. 3344 1. 3383 0. 0039	1. 3969 1. 4009 0. 0040	1, 4594 1, 4634 0, 0040	1. 5219 1. 5259 0. 0040	1. 5844 1. 5885 0. 0041	1. 6469 1. 6510 0. 0041	1. 7094 1. 7135 0. 0041	1. 7719 1. 7761 0. 0042	1. 8344 1. 8386 0. 0042	1. 8969 1. 9011 0. 0042

See footnotes at end of table.

Table 31.—Limiting dimensions and tolerances, classes 2 and 3 fit, American National 16-pitch thread series—Continued

					Size (inches)	ches)	Marin management and management about			
Dimensions and folerances 1	2	21/16	21/8	23/16	21/4	25/16	23/8	27/16	21/2	25/8
Bolts and Screws [Max	Inches 2. 0000 1. 9910 0. 0090	Inches 2. 0625 2. 0535 0. 0090	Inches 2. 1250 2. 1160 0. 0090	Inches 2. 1875 2. 1785 0. 0090	Inches 2, 2500 2, 2410 0, 0090	Inches 2. 3125 2. 3035 0. 0090	Inches 2. 3750 2. 3660 0. 0090	Inches 2. 4375 2. 4285 0. 0090	Inches 2. 5000 2. 4910 0. 0090	Inches 2. 6250 2. 6160 0. 0090
Minor diameterMax 3	1. 9233	1. 9858	2, 0483	2, 1108	2. 1733	2, 2358	2, 2983	2.3608	2, 4233	2. 5483
Class 2, pitch diameter (for gen- Maxeral use) Tol	1. 9594 1. 9533 0. 0061	2. 0219 2. 0158 0. 0061	2. 0844 2. 0782 0. 0062	2. 1469 2. 1407 0. 0062	2. 2094 2. 2032 0. 0062	2, 2719 2, 2656 0, 0063	2. 3344 2. 3281 0. 0063	2. 3969 2. 3905 0. 0064	2. 4594 2. 4530 0. 0064	2. 5844 2. 5779 0. 0065
Class 3, pitch diameter Min Tol	1. 9594 1. 9551 0. 0043	2. 0219 2. 0176 0. 0043	2. 0844 2. 0801 0. 0043	2. 1469 2. 1426 0. 0043	2. 2094 2. 2050 0. 0044	2. 2719 2. 2675 0. 0044	2. 3344 2. 3300 0. 0044	2. 3969 2. 3924 0. 0045	2. 4594 2. 4549 0. 0045	2. 5844 2. 5799 0. 0045
NUTS AND TAPPED HOLES										
Major diameterMin 4	2. 0000	2, 0625	2, 1250	2, 1875	2, 2500	2, 3125	2, 3750	2, 4375	2. 5000	2. 6250
Minor diameter ${\rm Max}^{6}$.	1. 9323 1. 9403 0. 0080	1. 9948 2. 0028 0. 0080	2. 0573 2. 0653 0. 0080	2. 1198 2. 1278 0. 0080	2. 1823 2. 1903 0. 0080	2. 2448 2. 2528 0. 0080	2. 3073 2. 3153 0. 0080	2. 3698 2. 3778 0. 0080	2. 4323 2. 4403 0. 0080	2. 5573 2. 5653 0. 0080
Class 2, pitch diameter (for gen- Mineral use) Tol	1. 9594 1. 9655 0. 0061	2. 0219 2. 0280 0. 0061	2. 0844 2. 0906 0. 0062	2. 1469 2. 1531 0. 0062	2. 2094 2. 2156 0. 0062	2. 2719 2. 2782 0. 0063	2. 3344 2. 3407 0. 0063	2. 3969 2. 4033 0. 0064	2. 4594 2. 4658 0. 0064	2. 5844 2. 5909 0. 0065
Class 3, pitch diameter{Max	1. 9594 1. 9637 0. 0043	2. 0219 2. 0262 0. 0043	2. 0844 2. 0887 0. 0043	2. 1469 2. 1512 0. 0043	2. 2094 2. 2138 0. 0044	2. 2719 2. 2763 0. 0044	2. 3344 2. 3388 0. 0044	2. 3969 2. 4014 0. 0045	2. 4594 2. 4639 0. 0045	2. 5844 2. 5889 0. 0045

		The second second									- Consideration of the Constitution of the Con
					0.2	Size (inches)					
Dimensions and tolerances 1	234	27/8	က	31/8	3¼	338	31/2	358	334	378	4
BOLTS AND SCREWS [Max	Inches 2. 7500 2. 7410	Inches 2. 8750 2. 8660	Inches 3. 0000 2. 9910	Inches 3. 1250 3. 1160	Inches 3. 2500 3. 2410	Inches 3. 3750 3. 3660	Inches 3. 5000 3. 4910	Inches 3. 6250 3. 6160	Inches 3. 7500 3. 7410	3. 8750 3. 8660	Inches 4. 0000 3. 9910
Minor diameterMax.3	2. 6733		2. 9233		3, 1733	3, 2983	3. 4233	3. 5483	3. 6733	3. 7983	3, 9233
meter {	2. 7094 2. 7028 0. 0066	2. 8344 2. 8278 0. 0066	2. 9594 2. 9527 0. 0067	3. 0844 3. 0776 0. 0068	3. 2094 3. 2025 0. 0069	3. 3344 3. 3275 0. 0069	3. 4594 3. 4524 0. 0070	3. 5844 3. 5773 0. 0071	3. 7094 3. 7023 0. 0071	3. 8344 3. 8272 0. 0072	3. 9594 3. 9522 0. 0072
Class 3, pitch diameter- $\left\{ \begin{array}{ll} Max\\ Min\\ Tol \end{array} \right.$	2. 7094 2. 7048 0. 0046	2. 8344 2. 8298 0. 0046	2. 9594 2. 9547 0. 0047	3. 0844 3. 0797 0. 0047	3. 2094 3. 2046 0. 0048	3. 3344 3. 3296 0. 0048	3. 4594 3. 4545 0. 0049	3. 5844 3. 5795 0. 0049	3. 7094 3. 7044 0. 0050	3. 8344 3. 8294 0. 0050	3. 9594 3. 9543 0. 0051
NUTS AND TAPPED HOLES											
Major diameter Min.4	2. 7500	2.8750	3.0000	3, 1250	3, 2500	3, 3750	3, 5000	3. 6250	3. 7500	3.8750	4. 0000
Minor diameter $\left\{\begin{array}{l} \mathrm{Min}_{} \\ \mathrm{Max.}^5 \end{array}\right\}$	2. 6823 2. 6903 0. 0080	2. 8073 2. 8153 0. 0080	2. 9323 2. 9403 0. 0080	3. 0573 3. 0653 0. 0080	3. 1823 3. 1903 0. 0080	3. 3073 3. 3153 0. 0080	3. 4323 3. 4403 0. 0080	3. 5573 3. 5653 0. 0080	3. 6823 3. 6903 0. 0080	3. 8073 3. 8153 0. 0080	3. 9323 3. 9403 0. 0080
Class 2, pitch diameter [Min (for general use){ Max	2. 7094 2. 7160 0. 0066	2. 8344 2. 8410 0. 0066	2. 9594 2. 9661 0. 0067	3. 0844 3. 0912 0. 0068	3. 2094 3. 2163 0. 0069	3. 3344 3. 3413 0. 0069	3. 4594 3. 4664 0. 0070	3. 5844 3. 5915 0. 0071	3. 7094 3. 7165 0. 0071	3. 8344 3. 8416 0. 0072	3. 9594 3. 9666 0. 0072
Class 3, pitch diameter- $\left\{ egin{array}{ll} \mathrm{Min}_{} \end{array} \right.$	2. 7094 2. 7140 0. 0046	2. 8344 2. 8390 0. 0046	2. 9594 2. 9641 0. 0047	3. 0844 3. 0891 0. 0047	3. 2094 3. 2142 0. 0048	3. 3344 3. 3392 0. 0048	3. 4594 3. 4643 0. 0049	3. 5844 3. 5893 0. 0049	3. 7094 3. 7144 0. 0050	3. 8344 3. 8394 0. 0050	3. 9594 3. 9645 0. 0051

¹ Pitch-diameter tolerances include errors of lead and angle. The class 2 tolerances are based on formulas in table 116 and a length of engagement of 6 threads or % inch. The class 3 tolerances are 70 percent of the class 2 tolerances. The %-inch size being in the American National fine-thread series, the tolerance for this size corresponds to that

Standard-size screw and nut of the American National fine-thread series.

5 Dimensions given for the maximum minor diameter of the screw are figured to the intersection of the worn-tool are with a center line through crest and root. The minimum minor diameter of the screw shall be that corresponding to a flat at the minor diameter.

of the minimum screw equal to j\$Xp, and may be determined by subtracting 0.0406 inch the minimum pitch diameter of the screw.

4 Dimensions for the minimum major diameter of the nut correspond to the basic flat (j\$Xp) and the profile at the major diameter of the nut shall be that corresponding the basic outline. The maximum major diameter of the nut shall be that corresponding to a flat at the major diameter of the maximum nut equal to j\$xXp, and may be determined by adding 0.0496 inch for the maximum pitch diameter of the nut.

§ Present Army ordinance practice follows Handbook H25 and the mineographed Supplement to Handbook H25 in the mineographed

AMERICAN NATIONAL EXTRA-FINE-THREAD SERIES FORM OF THREAD

10. The American National form of thread profile as specified in paragraphs 7 to 7f shall be used.

THREAD SERIES

10a. The American National extra-fine-thread series is intended for special uses where (1) thin-walled material is to be threaded, (2) thread depth of nuts clearing ferrules, coupling flanges, etc., must be held to a minimum, and (3) a maximum practicable number of threads are required within a given thread length. This thread series is the same as the SAE extra-fine-thread series, but it includes additional sizes. The nominal sizes and basic dimensions are specified in table 34. Limiting dimensions and tolerances for classes 2 and 3 fits are specified in table 35.

Table 34.—American National extra-fine-thread series

	Basic area of section at root of thread, $\pi K^2/4$	Sq. in. 0.0344 0.0581 0.0581 1201 1616 2.203 2.203 2.256 3.3685 4.388 4.388 6.5979 6.866 7.702 8.705 1.0895	1. 2082 1. 3330 1. 4640 1. 6011 1. 7444 1. 8937 2. 0493 2. 1873 2. 8917
	Helix angle at basic pitch diameter, s	deg min 257 1 579 1 386 1 1 22 1 1 25 1 1 6 1 1 6 1 1 0 0 57 0 55 0 55	0 48 0 445 0 0 40 0 0 38 0 37 0 35
ta	Minimum width of flat at major diameter of nut, p/24	Inch 0.00130 00130 00130 00149 00174 00174 00208 00208 00208 00208 00208 00208 00208 00231	. 00231 . 00231 . 00231 . 00231 . 00231 . 00231 . 00260
Thread data	Basic width of flat, p/8	$\begin{array}{c} Inch \\ 0.00391 \\ 0.00391 \\ 0.00446 \\ 0.00446 \\ 0.00521 \\ 0.00521 \\ 0.00525 \\ 0.00625 \\ 0.00625 \\ 0.00625 \\ 0.00625 \\ 0.00694 \\ 0.$. 00694 . 00694 . 00694 . 00694 . 00694 . 00694 . 00694
	Depth of thread, h	0.02030 0.02030 0.02030 0.02320 0.02320 0.02706 0.02706 0.02708 0.03248 0.03248 0.03248 0.03248 0.03248	. 03608 . 03608 . 03608 . 03608 . 03608 . 03608 . 04059
	Pitch, p	$^{Inch}_{0.03125}$ $^{0.03125}_{0.03125}$ $^{0.03125}_{0.03271}$ $^{0.03571}_{0.03571}$ $^{0.04167}_{0.04167}$ $^{0.05000}_{0.05000}$ $^{0.05000}_{0.0500}$ $^{0.05000}_{0.05556}$ $^{0.05556}_{0.05556}$. 05556 . 05556 . 05556 . 05556 . 05556 . 05556 . 06250
	Metric equivalent of major diameter		33. 338. 34. 925 36. 513 38. 100 39. 688 41. 275 42. 863 44. 450 50. 800
Basic diameters	$\begin{array}{c} \text{Minor} \\ \text{diameter,} \\ \end{array}$	Inches 0.2094 2.2719 2.3719 2.3214 2.3911 2.5084 5.509 6.830 6.830 7.475 8.8100 8.8725 9.830 9.830 1.0528 1.1153	1. 2403 1. 3028 1. 3653 1. 4278 1. 4903 1. 5528 1. 6153 1. 6688
	$\begin{array}{c} \text{Pitch} \\ \text{diameter,} \\ E \end{array}$	Imakes 0.2297 32922 35922 3547 4143 4768 5979 6604 7177 7800 8425 9050 9050 1.0889 1.1514 1.2139	1. 2764 1. 3389 1. 4014 1. 4639 1. 5264 1. 5889 1. 6514 1. 7094 1. 9594
B	$\begin{array}{c} \operatorname{Major} \\ \operatorname{diameter}, \\ D \end{array}$	7mthes 0.2500 3.3125 3.3125 4.375 5.000 5.625 6.875 7.500 8.875 1.0000 1.0000 1.1250 1.1250 1.250	1. 3125 1. 3750 1. 4375 1. 5000 1. 5625 1. 6250 1. 6875 1. 7500 2. 0000
	Threads per inch	222288 444400 22288 8811 8818	18 18 18 18 18 16 16
Identification	Size	Inches	
		3,76 1,76	15/16- 13/2- 17/16- 19/16- 11/16- 13/4-

Table 35.—Limiting dimensions and tolerances, classes 2 and 3 fits, American National extra-fine thread series

							Size (inch)						
. Dimensions and tolerances $^{\mathrm{1}}$	**	5/16	3%	5/16	22	%16	8%	11/16	3%	13/16	8%	15/16	1
3						Th	Threads per inch	ch		-			
	32	32	32	58	28	24	24	24	30	20	20	20	20
Bolts and Screws Classes 2 and 3, [Max major diame- [Min ter]	$\begin{array}{c} Inch \\ 0.2500 \\ 0.2446 \\ 0.0054 \end{array}$	$^{Inch}_{0.3125}_{0.3071}_{0.0054}$	$\begin{array}{c} Inch \\ 0.3750 \\ 0.3696 \\ 0.0054 \end{array}$	$^{Inch}_{0.4375}_{0.4313}_{0.0062}$	Inch 0. 5000 0. 4938 0. 0062	Inch 0. 5625 0. 5559 0. 0066	I_{nch}^{Inch} 0. 6250 0. 6184 0. 0066	$^{Inch}_{0.6875}_{0.6809}_{0.0066}$	$^{Inch}_{0.7500}_{0.7428}_{0.0072}$	$\begin{array}{c} Inch \\ 0.8125 \\ 0.8053 \\ 0.0072 \end{array}$	0.8750 0.8678 0.0072	$^{Inch}_{0.9375}_{0.9303}_{0.0072}$	$^{Inch}_{0.9928}_{0.0072}$
Classes 2 and 3, minor diameter. Max.2	0. 2117	0. 2742	0. 3367	0. 3937	0. 4562	0. 5114	0. 5739	0. 6364	0. 6887	0. 7512	0.8137	0. 8762	0. 9387
Class 2, pitch $\begin{cases} Max \\ Min.^{4} - \\ Tol \end{cases}$	0. 2297 0. 2265 0. 0032	0. 2922 0. 2889 0. 0033	0.3547 0.3513 0.0034	0. 4143 0. 4107 0. 0036	0. 4768 0. 4731 0. 0037	$\begin{array}{c} 0.5354 \\ 0.5314 \\ 0.0040 \end{array}$	$\begin{array}{c} 0.5979 \\ 0.5938 \\ 0.0041 \end{array}$	0. 6604 0. 6563 0. 0041	0. 7175 ,0. 7129 0. 0046	0. 7800 0. 7754 0. 0046	0. 8425 0. 8378 0. 0047	0. 9050 0. 9003 0. 0047	0. 9675 0. 9627 0. 0048
Class 3, pitch $\begin{cases} Max \\ Min.^{4-} \\ Tol \end{cases}$	0. 2297 0. 2275 0. 0022	0. 2922 0. 2899 0. 0023	$\begin{array}{c} 0.\ 3547 \\ 0.\ 3523 \\ 0.\ 0024 \end{array}$	$\begin{array}{c} 0.\ 4143 \\ 0.\ 4118 \\ 0.\ 0025 \end{array}$	0. 4768 0. 4742 0. 0026	0. 5354 0. 5326 0. 0028	0. 5979 0. 5950 0. 0029	0. 6504 0. 6575 0. 0029	0. 7175 0. 7143 0. 0032	0. 7800 0. 7768 0. 0032	0.8425 0.8392 0.0033	0. 9050 0. 9017 0. 0033	0. 9675 0. 9641 0. 0034
NUTS AND TAPPED HOLES													
Classes 2 and 3, major diameter_Min.³	0. 2500	0. 3125	0.3750	0. 4375	0. 5000	0. 5625	0. 6250	0. 6875	0.7500	0.8125	0.8750	0. 9370	1, 0000
Classes 2 and 3, $\begin{cases} Min_{} \\ Max_{} \end{cases}$ minor diameter- $\begin{cases} Tol_{} \end{cases}$	0. 2162 0. 2208 0. 0046	0. 2787 0. 2833 0. 0046	$\begin{array}{c} 0.3412 \\ 0.3458 \\ 0.0046 \end{array}$	0. 3988 0. 4041 0. 0053	0. 4613 0. 4666 0. 0053	0. 5174 0. 5235 0. 0061	0. 5799 0. 5860 0. 0061	0. 6424 0. 6485 0. 0061	0. 6959 0. 7027 0. 0068	0. 7584 0. 7652 0. 0068	0. 8209 0. 8277 0. 0068	0.8834 0.8902 0.0068	0. 9459 0. 9527 0. 0068
Class 2, pitch di- Max.4- ameter	0. 2297 0. 2329 0. 0032	0. 2922 0. 2955 0. 0033	$\begin{array}{c} 0.3547 \\ 0.3581 \\ 0.0034 \end{array}$	0. 4143 0. 4179 0. 0036	0. 4768 0. 4805 0. 0037	0. 5354 0. 5394 0. 0040	$\begin{array}{c} 0.5979 \\ 0.6020 \\ 0.0041 \end{array}$	0. 6604 0. 6645 0. 0041	0. 7175 0. 7221 0. 0046	0. 7800 0. 7846 0. 0046	0. 8425 0. 8472 0. 0047	0. 9050 0. 9097 0. 0047	0. 9675 0. 9723 0. 0048
Class 3, pitch di- Min ameter Tol	0. 2297 0. 2319 0. 0022	0. 2922 0. 2945 0. 0023	$\begin{array}{c} 0.3547 \\ 0.3571 \\ 0.0024 \end{array}$	$\begin{array}{c} 0.\ 4143 \\ 0.\ 4168 \\ 0.\ 0025 \end{array}$	0. 4768 0. 4794 0. 0026	$\begin{array}{c} 0.5354 \\ 0.5382 \\ 0.0028 \end{array}$	0. 5979 0. 6008 0. 0029	0. 6604 0. 6633 0. 0029	0. 7175 0. 7207 0. 0032	0. 7800 0. 7832 0. 0032	0. 8425 0. 8458 0. 0033	0. 9050 0. 9083 0. 0033	0. 9675 0. 9709 0. 0034

138 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	
135 1. 1250 14 1. 0889 15 1. 1250 16 1. 0568 17 1. 0889 18 1. 0568 19 1. 0568 10 1. 0724 10 0889 10 0889 10 0889 10 0889 11 0889 11 0889 11 0889 12 0. 0055 13 0. 0055 14 0. 0889 15 0. 0055 16 0. 0088 17 0. 0088 18 0. 0055 19 0. 0055 10 0889 10 0889	$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$
136 1376 134 1576 1 1576 1 1 1 1 1 1 1 1 1	$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$
136 1376 1376 1376 1376 1376 1376 13775 1377	$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$
138 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1946 1146 1146 1146 1146 1168 1.00542 1.00582 1.00582 1.00582 1.00582 1.00582 1.00582 1.00582 1.00582 1.00582 1.00582 1.00582 1.00583 1.00582 1.00583 1.00582 1.00592 1.00592 1.00592 1.005925 1.00592
	$\begin{array}{ c c c c c c c c c c c c c c c c c c c$

 Pitch diameter tolerances include errors of lead and angle. The class 2 tolerances are based on the formulas in table 116 and a length of engagement of 6 threads. The class 3 folerances are 70 percent of the class 2 tolerances.

³ Dimensions for the minimum major diameter of the nut correspond to the basic flat $(\frac{3}{4}X_{2})_{2}$, and the profile at the major diameter produced by a worn tool must not fall below the basic outline. The maximum major diameter of the nut shall be that corresponding to a flat at the major diameter of the maximum nut equal to $\frac{1}{2}4 \times X_{2}$, and may be determined by adding $\frac{1}{2}5 \times \hbar$ (or 0.7399 g) to the maximum pitch diameter of the nut.

These dimensions are the minimum metal or "not go" size. The "go," or basic size is the one that should be placed on the component drawing with the tolerance.

SIZES OF TAP DRILLS

11. The essential requirement of a tap drill is that the hole produced by it shall be such that, when tapped with a screw thread, the minor diameter of the tapped hole shall be within the specified limits. It should be noted that the minor diameters of the tapped holes are

the same for classes 1 to 4, inclusive.

11a. If the drill is too large, the minor diameter of the tapped hole will also be too large, and the thread in the nut will be too shallow, that is, too small a percentage of a full thread. As an extreme case, the threads in the tapped hole will engage only the tops of the threads on a screw of correct size, and under stress the threads of the screw will strip and the full strength of the fastening will not be developed.

11b. If, on the other hand, the tap drill is too small, the tap will be forced to cut a thread of full depth, and in the extreme case to act as a reamer also. This will result in excessive power consumption and tap breakage, and will also make the minor diameter of the tapped hole dependent upon the minor diameter of the tap. This is undesirable, since the minor diameter of the tap is not, in general, held to the same close limits as the other tap elements, and as a result the minor diameter of a hole tapped under these conditions may be in error even

though the tap is otherwise correct.

11c. It is a well-known fact that the size of the hole produced by a tap drill depends to some extent upon the method of grinding the drill, the material drilled, the lubricant used, and the alinement, speed, and feed of operation. This being true, it is apparent that fixing the diameter of the tap drill does not completely fix the diameter of the drilled hole. The most that can be accomplished is to fix the drill diameters between certain limits and to depend upon correct grinding, lubrication, and operation to keep the diameter of the holes

within prescribed limits.

11d. There are listed in tables 122 and 123 from Handbook H28, and in the additional tables 123 (A) through 123 (D), all drill sizes regularly carried in stock, both English and metric, which fall between the limiting dimensions of the minor diameter of the threaded hole for the American National coarse-, fine-, extra-fine-, and the 8-, 12-, and 16-pitch-thread series. There are several thread sizes, however, for which there are no stock drills falling within the minor diameter limits, and for these the nearest drills outside of the maximum and minimum limits are listed in italics. If the material to be tapped is such that there is considerable "spin-up" on minor diameter during tapping, then the larger of the two drills listed for a given size should be selected. If the material is cast iron or other material with little or no "spin-up", then the smaller of the two drills listed should be chosen. It will usually cut oversize by a sufficient amount to bring the minor diameter above the minimum limit.

Table 122 .- Sizes of tap drills [American National coarse-thread series]

		Mino	r diameter o		Stock drills correspondence of ba	nding to 100 sic thread de	percent to 50
Size of thread	Threads per inch	Basic	Maximum 1	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
1	64	Inch 0. 0527	Inch 0. 0623	Inch 0. 0561	1.45 mm 1.50 mm 1.55 mm	Inch 0. 0571 . 0591 . 0610	78 68 59
2	56	. 0628	. 0737	. 0667	\{\#51 \#50 \#49	. 0670 . 0700 . 0730	82 69 56
3	48	. 0719	. 0841	. 0764	\$\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	. 0781 . 0810 . 0827	77 67 60
4	40	. 0795	. 0938	. 0849	#44	. 0860 . 0890 . 0906 . 0937	80 71 66 56
5	40	. 0925	. 1062	. 0979	#39 #38 ³ 2.60 mm #37	. 0995 . 1015 . 1024 . 1040	79 72 70 65
6	32	. 0974	. 1145	. 1042	\begin{cases} \#36_\\ \%4 \text{ in}\\ \#33\\ \\ \end{cases}	. 1065 . 1094 . 1130	78 70 62
-8	32	. 1234	. 1384	. 1302	3.40 mm #29 3.50 mm	. 1339 . 1360 . 1378	74 69 65
10	24	. 1359	. 1559	. 1449	{#26 #24	. 1470 . 1520	79 70
12	24	. 1619	. 1801	. 1709	1½4 in #17 #16 #15	. 1719 . 1730 . 1770 . 1800	82 79 72 67
1/4	20	. 1850	. 2060	. 1959	#8	. 1960 . 1990 . 2031	83 79 72
5/16	18	. 2403	. 2630	. 2524	{F	. 2570 . 2610	77 71
3/8	16	. 2938	. 3184	. 3073	{5/16 in	. 3125 . 3160	77 73
% 6	14	. 3447	. 3721	. 3602	U	. 3680	75
1/2	13	. 4001	. 4290	. 4167	² 7/ ₆₄ in	. 4219	78
%16	12	. 4542	. 4850	. 4723	31/64 in	. 4844	72

¹ Present Army Ordnance practice follows Handbook H25 and the mimeographed Supplement to Handbook H28 in the maximum minor diameters of nuts.

⁸ See footnote at end of table 123 (D).

Table 122.—Sizes of tap drills—Continued

[American National coarse-thread series]

		Mino	or diameter of	of nut	Stock drills correspondence of base	nding to 100 sic thread de	percent to 50 epth
Size of thread	Threads per inch	Basic	Maximum 1	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
5/8	11	Inch . 5069	Inch . 5397	Inch . 5266	\begin{cases} \{^{1}\%_{32}\text{ in}_{} \\ 13.5 \text{ mm}_{} \end{cases}	Inch . 5312 . 5315	79
3/4	10	. 6201	. 6553	. 6417	16.5 mm	. 6496	77
7/8	9	. 7307	. 7689	. 7547	4%4 in 19.5 mm	. 7656 . 7677	76 74
1	8	. 8376	. 8795	. 8647	22 mm ½ in	. 8661 . 8750	82 77
11/8	7	. 9394	. 9858	. 9704	25 mm 63/64 in	. 9842 . 9844	76 76
11/4	7	1. 0644	1. 1108	1. 0954	28 mm 1½ in	1. 1024 1. 1094	80 76
13/8	6	1. 1585	1. 2126	1. 1946	30.5 mm 113/64 in	1. 2008 1. 2031	80
1½	6_	1. 2835	1. 3376	1. 3196	121/64 in	1. 3281	79
13/4	5	1. 4902	1. 5551	1. 5335	39 mm 1 ³⁵ ⁄ ₆₄ in 39.5 mm	1. 5354 1. 5469 1. 5551	88 78 78
2	$4\frac{1}{2}$	1. 7113	1. 7835	1. 7594	$\begin{cases} 1^{4}\%_{4} \text{ in}_{12}\%_{32} \text{ in}_{12}\%_{32} \text{ in}_{12}\%_{32} \end{cases}$	1. 7656 1. 7716 1. 7812	81 79 76
$2\frac{1}{4}$	4½	1. 9613	2. 0335	2. 0094	$\begin{cases} 2\frac{1}{64} & \text{in} \\ 51.5 & \text{mm} \\ 2\frac{1}{32} & \text{in} \\ \dots \end{cases}$	2. 0156 2. 0276 2. 0312	81 77 76
$2\frac{1}{2}$	4	2. 1752	2. 2564	2. 2294	$\begin{cases} 2^{15}/_{4} \text{ in}_{} \\ 57 \text{ mm}_{} \\ 2^{1}/_{4} \text{ in}_{} \end{cases}$	2. 2344 2. 2441 2. 2500	82 79 77
2¾	4	2. 4252	2. 5064	2. 4794	63 mm	2. 4803 2. 4844 2. 5000 2. 5000	83 82 77 77
3	4	2. 6752	2. 7564	2. 7294	24764 in 69.5 mm 234 in 70 mm	2. 7344 2. 7362 2. 7500 2. 7559	82 81 77 75
31/4	4	2. 9252	3. 0064	2. 9794	$\begin{cases} 2^{63}/_{64} \text{ in} \\ 76 \text{ mm} \\ 3 \end{cases}$	2. 9844 2. 9921 3. 0000	82 79 77
$3\frac{1}{2}$	4	3. 1752	3. 2564	3. 2294	3¼ in	3. 2500	77
3¾	4	3. 4252	3. 5064	3. 4794	3½ in	3. 5000	77

Table 123 .- Sizes of tap drills [American National fine-thread series]1

_			American	autonai nne-	thread series;		
	Mhuanda.	Mino	r diameter o	of nut	Stock drills correspondence percent of ba	nding to 100 p sic thread de	percent to 50
Size of thread	Threads per inch	Basic	Maximum ²	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
		Inch	Inch	Inch		Inch	
0	80	0. 0438	0. 0514	0. 0465	3%4 in 1.25 mm	0. 0469	81 67
1	72	. 0550	. 0634	. 0580	1.50 mm 1.55 mm	. 0591 . 0610	77 67
2	64	. 0657	. 0746	. 0691	 #49	. 0700 . 0730	79 64
3	56	. 0758	. 0856	. 0797	\begin{cases} #46 2.10 \text{ mm} #44	. 0810 . 0827 . 0860	78 70 56
4	48	. 0849	. 0960	. 0894	2.30 mm 3/32 in #41	. 0906 . 0937 . 0960	79 68 59
5	44	. 0955	. 1068	. 1004	2.60 mm #37 #36	. 1024 . 1040 . 1065	77 71 63
6	40	. 1055	. 1179	. 1109	{#33 #32	. 1130 . 1160	77 68
					(3.40 mm	. 1339	83
8	- 36	. 1279	. 1402	. 1339	3.50 mm %4 in	. 1360 . 1378 . 1406	78 73 65
10	32	. 1494	. 1624	. 1562	\$\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	. 1562 . 1590 . 1610 . 1660	83 76 71 59
12	28	. 1696	. 1835	. 1773	#15 4.70 mm #13 _{3/6} in	. 1800 . 1850 . 1875	78 67 61
1/4	28	. 2036	. 2173	. 2113	#3	. 2130	80
5∕16f_	24	. 2584	. 2739	. 2674	{17/64 in	. 2656 . 2720	87 75
3/8	24	. 3209	. 3364	. 3299	Q	. 3320	79

Drill sizes up to ½ inch are in agreement with ASA B5.12—1940, Twist Drills, Straight Shank, published by the ASME, 29 West 39th Street, New York, N. Y.
 Present Army Ordnance practice follows NBS Handbook H25 and the mimeographed Supplement to Handbook H28 in the maximum minor diameters of nuts.
 See footnote at end of table 123 (D).

Table 123.—Sizes of tap drills—Continued

[American National fine-thread series]

			[IIIIIIIIIIIIIIIIIIIIIIIIIIIIIIIIIIIII				
Size of	Threads	Mino	r diameter o	of nut	Stock drills correspon percent of ba	nding to 100 p sic thread de	percent to 50
thread	per inch	Basic	Maximum	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
7/16-	20	Inch 0. 3725	Inch 0. 3906	Inch 0. 3834	{W	Inch 0. 3860 . 3906	. 79 72
1/2	20	. 4350	. 4531	. 4459	²⁹ / ₆₄ in	. 4531	72
%16	18	. 4903	. 5100	. 5024	0.5062	. 5062	78
5/8	18	. 5528	. 5725	. 5649	14.5 mm	. 5709	75
3/4	16	. 6688	. 6903	. 6823	17.5 mm	. 68 7 5 . 6890	77 75
7/8	14	. 7822	. 8062	. 7977	5½ in 20.5 mm	. 7969 . 8071	84 73
1	14	. 9072	. 9312	. 9227	23.5 mm	. 9252	81
11/8_	12	1. 0167	1. 0438	1. 0348	26.5 mm	1. 0433	75
11/4_	12	1. 1417	1. 1688	1. 1598	29.5 mm	1. 1614	82
13/8_	12	1. 2667	1. 2938	1. 2848	{%2 in 1964 in	1. 2812 1. 2969	8 72
1½-	12	1. 3917	1. 4188	1. 4098	36 mm	1. 4173	76

Table 123 (A).—Sizes of tap drills [American National 8-pitch-thread series]

		Min	or diameter o	of nut	Stock drills and corr basic th	esponding p read depth	ercentage of
Size of thread	Threads per in.	Basic	Maximum ¹	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
1	8	Inch 0. 8376	Inch 0. 8795	Inch 0. 8647	{22 mm 7/8 in	Inch 0. 8661 . 8750	82 77
11/8	8	. 9626	1. 0045	. 9897	{1 in 25.5 mm	1. 0000 1. 0039	77 75
11/4	8	1. 0876	1. 1295	1. 1147		1. 1220 1. 1250	79 77
13%	8	1. 2126	1. 2545	1. 2397	{31.5 mm	1. 2402 1. 2500	83 77
1½	8	1. 3376	1. 3795	1. 3647	{1% in 35 mm	1. 3750 1. 3780	77 75
15/8	8	1. 4626	1. 5045	1. 4897	{38 mm	1. 4961 1. 5000	79 77
1¾	8	1. 5876	1. 6295	1. 6147	1% in	1. 6250	77
1 1/8	8	1. 7126	1. 7545	1. 7397	{1¾ in 44.5 mm	1. 7500 1. 7520	77 76
2	8	1. 8376	1. 8795	1. 8647	{47.5 mm 1½ in	1. 8701 1. 8750	80 77
21/8	8	1. 9626	2. 0045	1. 9897	2 in	2. 0000	77
21/4	8	2. 0876	2. 1295	2. 1147	${2\frac{1}{8} \text{ in}}_{54 \text{ mm}}$	2. 1250 2. 1260	77 76
2½	8	2. 3376	2. 3795	2. 3647	23/8 in	2. 3750	77
2¾	8	2. 5876	2. 6295	2. 6147	${66.5 \text{ mm} _ _ _ \atop 2\frac{5}{8} \text{ in} _ _ _ _}$	2. 6181 2. 6250	81 77
3	8	2. 8376	2. 8795	2. 8647	{73 mm 2½ in	2. 8740 2. 8750	78 77
31/4	8	3. 0876	3. 1295	3. 1147	31/8 in	3. 1250	77
3½	8	3. 3376	3. 3795	3. 3647	33/8 in	3. 3750	77

 $^{^{\}rm I}$ Present Army Ordnance practice follows Handbook H25 and the mimeographed Supplement to Handbook H28 in the maximum minor diameters of nuts.

Table 123 (B).—Sizes of tap drills

[American National 12-pitch thread series]

-				- Pitol	thread series		
		Min	or diameter o	of nut	Stock drills and corr basic th	esponding p read depth ¹	ercentage of
Size of thread	Threads per in.	Basic	Maximum	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
		Inch	Inch	Inch	(7)	Inch	
1/2	12	0. 3917	0. 4225	0. 4098	$\begin{cases} Z^3$	0. 4130 . 4134 . 4219	80 80 72
%16	12	. 4542	. 4850	. 4723	$\begin{cases} 12 \text{ mm}^3 \\ 31/64 \text{ in} \\ \dots \end{cases}$. 4724 . 4844	83 72
5/8	12	. 5167	. 5438	. 5348	{13.5 mm 35/64 in	. 5315 . 5469	86 72
11/16	12	. 5792	. 6063	. 5973	$\begin{cases} 19/32 \ in_{} \\ 39/64 \ in_{} \end{cases}$. 5938 . 6094	87 72
3/4	12	. 6417	. 6688	. 6598	$ \begin{cases} 21/32 \ in_{} \\ 17 \ mm_{} \end{cases} $. 6562 . 6693	87 75
13/16	12	. 7042	. 7313	. 7223	18.5 mm	. 7283	78
7/8	12	. 7667	. 7938	. 7848	20 mm	. 7874	81
15/16	12	. 8292	. 8563	. 8473	\{21.5 mm \{55/64 in	. 8465 . 8594	84 72
1	12	. 8917	. 9188	- 9098	$ \begin{cases} 29/32 \ in_{} \\ 59/64 \ in_{} \end{cases} $. 9062 . 9219	87 72
1½6	12	. 9542	. 9813	. 9723	$ \begin{cases} 31/32 \ in_{} \\ 25 \ mm_{} \end{cases} $. 9687 . 9843	87 72
11/8	12	1. 0167	1. 0438	1. 0348	26.5 mm	1. 0433	75
13/16	12	1. 0792	1. 1063	1. 0973	28 mm	1. 1024	79
11/4	12	1. 1417	1. 1688	1. 1598	29.5 mm	1. 1614	82
15/16	12	1. 2042	1. 2313	1. 2223	\{31 mm 1 15/64 in	1. 2205 1. 2344	85 72
13/8	12	1. 2667	1. 2938	1. 2848	{1 9/32 in 1 19/64 in	1. 2812 1. 2969	87 72
17/16	12	1. 3292	1. 3563	1. 3473	$ \begin{cases} 1 & 11/32 \ in _ _ _ \\ 34.5 & mm _ _ _ _ \end{cases} $	1. 3438 1. 3583	87 73

 $^{^{\}rm l}$ Sizes in italics are not within the specified limits for minor diameter of nut. $^{\rm l}$ See footnote at end of table 123 (D).

Table 123 (B).—Sizes of tap drills—Continued
[American National 12-pitch thread series]

		Mine	or diameter o	f nut	Stock drills and correbasic th	esponding p read depth 1	ercentage of
Size of thread	Threads per in.	Basic	Maximum	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
1½	12	Inch 1. 3917	Inch 1. 4188	Inch 1. 4098	36 mm	Inch 1. 4173	76
15/8	12	1. 5167	1. 5438	1. 5348	39 mm	1. 5354	83
1¾	12	1. 6417	1. 6688	1. 6598	$ \begin{cases} 1 & 21/32 \ in_{} \\ 1 & 43/64 \ in_{} \end{cases} $	1. 6562 1. 6719	-87 72
17/8	12	1. 7667	1. 7938	1. 7848	45.5 mm	1. 7913	77
2	12	1. 8917	1. 9188	1. 9098	{48.5 mm 1 59/64 in	1. 9094 1. 9219	84 72
21/8	12	2. 0167	2. 0438	2. 0348	{2 1/32 in 2 3/64 in	. 2. 0312 2. 0469	87 72
21/4	12	2. 1417	2. 1688	2. 1598	55 mm	2. 1654	78
23/8	12	2. 2667	2. 2938	2. 2848	{58 mm 2 19/64 in	2. 2835 2. 2969	85 72
2½	12	2. 3917	2. 4188	2. 4098	{2 13/32 in 61.5 mm	2. 4062 2. 4213	87 73
25/8	12	2. 5167	2. 5438	2. 5348	64.5 mm	2. 5394	79
2¾	12	2. 6417	2. 6688	2. 6598	\{67.5 mm 2 43/64 in	2. 6575 2. 6719	- 85 72
21/8	12	2. 7667	2. 7938	2. 7848	$ \begin{cases} 2 & 25/32 & in_{} \\ 71 & mm_{} \end{cases} $	2. 7812 2. 7953	87 74
3	12	2. 8917	2. 9188	2. 9098	74 mm	2. 9134	80
31/8	12	3. 0167	3. 0438	3. 0348	\[\begin{cases} 3 & 1/32 & in \ 3 & 1/16 & in \end{cases} \]	3. 0312 3. 0625	87 58
31/4	12	3. 1417	3. 1688	3. 1598	\[\begin{cases} 3 & 5/32 & in\\ 3 & 3/16 & in\end{cases} \]	3. 1562 3. 1875	87 58
33/8	12	3. 2667	3. 2938	3. 2848	\[\begin{cases} 3 & 9/32 & in \ \\ 3 & 5/16 & in \ \end{cases} \]	3. 2812 3. 3125	87 58
3½	12	3. 3917	3. 4188	3. 4098	3 7/16 in	3. 4375	58

¹ Sizes in italics are not within the specified limits for minor diameter of nut.

Table 123 (C).—Sizes of tap drills [American National 16-pitch-thread series]

	1	[27	morioan ivat		n-enread seriesj		
		Min	or diameter o	f nut	Stock drills and corr basic th	esponding p read depth	ercentage of
Size of thread	Threads per inch	Basic	Maximum ¹	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
3/4	16	Inch 0. 6688	Inch 0. 6903	Inch 0. 6823	\begin{cases} \frac{11}{16} \text{ in } \\ 17.5 \text{ mm} \\ \\ \end{cases} \]	Inch 0. 6875 . 6890	77 75
13/16	16	. 7313	. 7528	. 7448	{19 mm 3⁄4 in	. 7480 . 7500	79 77
7/8	16	. 7938	. 8153	. 8073	¹³ / ₁₆ in	. 8125	77
15/16	16	. 8563	. 8778	. 8698	7/8 in	. 8750	77
1	16	. 9188	. 9403	. 9323	¹⁵ / ₁₆ in	: 9375	77
11/16	16	. 9813	1. 0028	. 9948	1 in	1. 0000	77
11/8	16	1. 0438	1. 0653	1. 0573	{1½6 in 27 mm	1. 0625 1. 0630	77 76
13/16	16	1. 1063	1. 1278	1. 1198	{28.5 mm 1½ in	1. 1220 1. 1250	81 7 7
11/4	16	1. 1688	1. 1903	1. 1823	13/16 in	1. 1875	77
15/16	16	1. 2313	1. 2528	1. 2448	1¼ in	1. 2500	77
13/8	16	1. 2938	1. 3153	1. 3073	15/16 in	1. 3125	77
17/16	16	1. 3563	1. 3778	1. 3698	13% in	1. 3750	77
1½	16	1. 4188	1. 4403	1. 4323	36.5 mm 1½6 in	1. 4370 1. 4375	78 77
1%	16	1. 4813	1. 5028	1. 4948	38 mm 1½ in	1. 4961 1. 5000	82 77
15/8	16	1. 5438	1. 5653	1. 5573	1% in	1. 5625	. 77
111/16	16	1. 6063	1. 6278	1. 6198	1% in	1. 6250	77
13/4	16	1. 6688	1. 6903	1. 6823	111/16 in	1. 6875	77
113/16	16	1. 7313	1. 7528	1. 7448	{1¾ in 44.5 mm	1. 7500 1. 7520	77 75

 $^{^{1}}$ Present Army ordnance practice follows Handbook H25 and the mimeographed Supplement to Handbook H28 in the maximum minor diameters of nuts.

Table 123 (C).—Sizes of tap drills—Continued

[American National 16-pitch-thread series]

		Min	or diameter o	of nut	Stock drills and corr basic th	esponding p read depth	ercentage of
Size of thread	Threads per in.	Basic	Maximum	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
		Inch	Inch	Inch	(46	Inch	70
1%	16	1. 7938	1. 8153	1. 8073	{46 mm 1 ¹³ / ₁₆ in	1. 8110 1. 8125	79 77
115/16	16	1. 8563	1. 8778	1. 8698	{47.5 mm 1½ in	1. 8701 1. 8750	83 77
2	16	1. 9188	1. 9403	1. 9323	115/16 in	1. 9375	77
2½6	16	1. 9813	2. 0028	1. 9948	2 in	2. 0000	77
21/8	16	2. 0438	2. 0653	2. 0573	2½ in	2. 0625	77
23/16	16	2. 1063	2. 1278	2. 1198	{2½ in 54 mm	2. 1250 2. 1260	77 76
21/4	16	2. 1688	2. 1903	2. 1823	$\begin{cases} 55.5 \text{ mm}_{} \\ 2\frac{3}{16} \text{ in}_{} \end{cases}$	2. 1850 2. 1875	80 77
25/16	16	2. 2313	2. 2528	2. 2448	2¼ in	2. 2500	77
23/8	16	2. 2938	2. 3153	2. 3073	25/16 in	2. 3125	77
27/16	16	2. 3563	2. 3778	2. 3698	23/8 in	2. 3750	77
2½	16 .	2. 4188	2. 4403	2. 4323	21/16 in	2. 4375	77
2 %	16	2. 5438	2. 5653	2. 5573	$\begin{cases} 65 \text{ mm}_{} \\ 2\%_{16} \text{ in}_{} \end{cases}$	2. 5590 2. 5625	81 77
2¾	16	2. 6688	2. 6903	2. 6823	211/16 in	2. 6875	77
21/8	16	2. 7938	2. 8153	2. 8073	$\begin{cases} 2^{13}/_{16} \text{ in}_{} \\ 71.5 \text{ mm}_{} \end{cases}$	2. 8125 2. 8150	77 74
3	16	2. 9188	2. 9403	2. 9323	$ \begin{cases} 74.5 \text{ mm}_{} \\ 2^{15}/_{16} \text{ in}_{} \end{cases} $	2. 9331 2. 9375	82 77
31/8	16	3. 0438	3. 0653	3. 0573	3½ in	3. 0625	77
31/4	` 16	3. 1688	3. 1903	3. 1823	33/16 in	3. 1875	77
33/8	16	3. 2938	3. 3153	3. 3073	35/16 in	3. 3125	77
3½	16	3. 4188	3. 4403	3. 4323	31/16 in	3. 4375	77

Table 123 (D).—Sizes of tap drills [American National extra-fine-thread series]

		Min	or diameter o	f nut	Stock drills and corr basic th	esponding p read depth 1	ercentage of
Size of thread	Threads per inch	Basic	Maximum ²	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
		Inch	Inch	Inch	(5.5 mm.³	Inch 0. 2165	09
1/4	32	0. 2094	0. 2208	0. 2162	7/32 in 5.6 mm.3	. 2188	83 77 73
5/16	32	. 2719	. 2833	. 2787	$\begin{cases} 7.1 \text{ mm.}^3 _ _ _ \\ \text{K}^3 _ _ _ _ \\ \frac{9}{32} \text{ in} _ _ _ \end{bmatrix}$. 2795 . 2810 . 2812	81 77 77
3/8	32	. 3344	. 3458	. 3412	8.7 mm. ³ 1 ¹ / ₃₂ in 8.75 mm. ³	. 3425 . 3438 . 3445	80 77 75
7/16	28	. 3911	. 4041	. 3988	$\begin{cases} X_{} \\ Y^{3}_{} \end{cases}$. 3970 . 4040	87 72
1/2	. 28	. 4536	. 4666	. 4613	$\begin{cases} ^{29}_{64} in_{} \\ ^{15}_{32} in_{} \end{cases}$. 4531 . 4687	101 67
9/16	24	. 5084	. 5235	. 5174	$\begin{cases} {}^{3}{}^{3}{}^{6}{}_{4} & in_{} \\ {}^{1}{}^{7}{}^{3}{}_{2} & in_{} \end{cases}$. 5156 . 5312	87 58
5/8	24	. 5709	5860	. 5799	$\begin{cases} 37/64 & in_{} \\ 15 & mm_{} \end{cases}$. 5781 . 5906	87 64
11/16	24	. 6334	. 6485	. 6424	\begin{cases} \b	6406 . 6496	87 70
3/4	20	. 6850	. 7027	. 6959	{17.5 mm	. 6890 . 7031	94 72
13/16	20	. 7475	. 7652	. 7584	\begin{cases} \frac{3\lambda}{4} in_{} \\ \frac{49\lambda}{64} in_{} \end{cases}	. 7500 . 7656	96 72
7/8	20	. 8100	. 8277	. 8209	21 mm	. 8268	74
15/16	20	. 8725	. 8902	. 8834	22.5 mm	. 8858	80
1	20	. 9350	. 9527	. 9459	24 mm	. 9449 . 9531	85 72
11/16	18	. 9903	1. 0099	1. 0024	25.5 mm	1. 0040	81
11/8	18	1. 0528	1. 0724	1. 0649	{27 mm 15/64 in	1. 0630 1. 0781	86 65
13/16	18	1. 1153	1. 1349	1. 1274	$\left\{ egin{array}{lll} 1 \ 1 \ 1 \ 1 \ 1 \ 1 \ 1 \end{array} \right.$	1. 1250. 1. 1406	87 65
11/4	18	1. 1778	1. 1974	1. 1899	$ \begin{cases} 1\% & in_{} \\ 30.5 & mm_{} \end{cases} $	1. 1875 1. 2008	87 68

Sizes in italics are not within the specified limits for minor diameter of nut.
 Present Army ordnance practice follows Handbook H25 and the mimeographed Supplement to Handbook H28 in the maximum minor diameters of nuts.
 These sizes are not included as standard in American Standard B 5.12-1940 for Twist Drills, Straight Shank, but are listed in the appendix thereto.

Table 123 (D).—Sizes of tap drills—Continued

[American National extra-fine-thread series]

	-	Mine	or diameter o	f nut	Stock drills and correbasic th	esponding p read depth	ercentage of
Size of thread	Threads per inch	Basic	Maximum	Minimum	Nominal size	Diameter	Percentage of depth of basic thread
15/16	18	Inch 1. 2403	Inch 1. 2599	Inch 1. 2524	32 mm	Inch 1. 2598	73
13/8	18	1. 3028	1. 3224	1. 3149	33.5 mm	1. 3189	78
17/16	18	1. 3653	1. 3849	1. 3774	35 mm	1. 3780	82
1½	18	1. 4278	1. 4474	1. 4399	$\begin{cases} 1\frac{7}{16} & in_{} \\ 1^{29}64 & in_{} \end{cases}$	1. 4375 1. 4531	87 65
1%6	18	1. 4903	1. 5099	1. 5024	\begin{cases} \begin{cases} 1\frac{1}{2} & in_{} \\ 1\frac{3}{64} & in_{} \end{cases} \end{cases}	1. 5000 1. 5156	87 65
1%	18	1. 5528	1. 5724	1. 5649	{1%6 in 40 mm	1. 5625 1. 5748	87 70
111/16	18	1. 6153	1. 6349	1. 6274	41.5 mm	1. 6339	74
134	16	1. 6688	1. 6903	1. 6823	111/16 in	1. 6875	77
2	16	1. 9188	1. 9403	1. 9323	115/16 in	1. 9375	77

LABELING

12. Where the dimensions are to be guaranteed, the following form of statement on labels, invoices, catalogues, etc., is recommended:

The _____ guarantees that for the respec-

tive classes of fit as identified or labeled, these screw threads conform to Commercial Standard CS24-43 as issued by the National Bureau of Standards of the U. S. Department of Commerce.

EFFECTIVE DATE

The standard is effective for new production from February 10, 1943.

STANDING COMMITTEE

The following individuals comprise the membership of the standing committee, which is to review, prior to circulation for acceptance, revisions proposed to keep the standard abreast of progress. Most organizations nominated their own representatives. Comment concerning the standard and suggestions for revision may be addressed to any member of the committee or to the Division of Trade Standards, National Bureau of Standards, which acts as secretary for the committee.

Manufacturers:

Carl W. Bettcher (Chairman), Eastern Machine Screw Corporation, New Haven, Conn.

J. J. Tomalis, American Screw Co., 21 Stevens Street, Providence, R. I.

George S. Case, Lamson & Sessions Co., 1975 W. 85th Street, Cleveland, Ohio-J. S. Davey, Russell, Burdsall & Ward Bolt & Nut Co., Port Chester, N. Y. J. H. Edmonds, Lebanon Plant, Bethlehem Steel Co., Lebanon, Pa. H. C. Erdman, National Screw & Mfg. Co., 2440 E. 75th St., Cleveland, Ohio W. C. Stewart, American Institute, Bolt, Nut and Rivet Mfrs., 1550 Hanna Bldg., Cleveland, Ohio.
F. P. Tisch, Pheoll Mfg. Co., 5700 Roosevelt Road, Chicago, Ill. Charles C. Winter, Winter Bros. Co., Wrentham, Mass.

Distributors:

G. Cheston Carey, Carey Machinery & Supply Co., 119 E. Lombard Street' Baltimore, Md.

H. H. SMITH, Strong, Carlisle & Hammond Co., 1392 W. 3d Street, Cleveland, Ohio.

Consumers:

W. B. Barth, General Motors Corporation, Standards Section, 15–158 General Motors Bldg., Detroit, Mich. Lt. Col. Harry B. Hambleton, Office of Chief of Ordnance, War Department,

Washington, D. C.

A. M. Houser, Crane Company, 836 S. Michigan Ave., Chicago, Ill. L. A. Wenn, International Business Machines Co., North Street, Endicott, N. Y. H. W. Samson, Standards Department, General Electric Co., Schenectady, N. Y. Lt. Comdr. J. W. Huckert, USN, Naval Gun Factory, U. S. Navy Yard, Washington, D. C.

Laboratories:

H. W. Bearce, Interdepartmental Screw Thread Committee, National Bureau of Standards, Washington, D. C. EARLE BUCKINGHAM, Massachusetts Institute of Technology, Cambridge, Mass.

HISTORY OF PROJECT

In the United States the standardization of screw threads was begun with the appointment of a special committee by the Franklin Institute on April 21, 1861, for the investigation of a proper system of screw threads, bolt heads, and nuts. From this beginning there was developed a system variously known as the Franklin Institute thread, the Sellers thread, or the United States thread. Later a system having finer pitches was recommended by the Society of Automotive Engineers, and a machine-screw-thread series providing smaller sizes of screws than the United States series was recommended by the Ameri-

can Society of Mechanical Engineers.

On July 18, 1918, the Congress authorized the appointment of the National Screw Thread Commission, consisting of nine members, to "ascertain and establish standards for screw threads" which when "accepted and approved shall be adopted and used in the several manufacturing plants under the control of the War and Navy Departments, and, so far as practicable, in all specifications for screw threads in proposals for manufactured articles, parts, or materials to be used under the direction of these departments." The National Screw Thread Commission issued printed reports in 1921, 1924, 1928, and 1933, based upon a long series of hearings and investigations both in the United States and abroad.

While the recommendations of the NSTC are mandatory upon the War and Navy Departments, and, as far as practicable, apply also to purchases by all Government departments, it seemed desirable to determine the extent to which these standards were being applied within the industries concerned. Accordingly, on May 8, 1929, the National Screw Thread Commission requested the cooperation of the National Bureau of Standards to determine the extent of adoption

and use of the NSTC recommendations in industry.

The hearing of the NSTC having performed all the essential functions of the general conferences normally required as a part of the procedure leading to the establishment of commercial standards, and the recommendations of the NSTC having attained national recognition and a large following, it seemed logical to proceed directly with the circulation of the essential screw-thread tables and tolerances to industry for written acceptance. This was done and resulted in the impressive roster of organizations, listed on page V of CS24–30 and CS25–30, which indicated in writing their intention of making the American National Standard Screw Threads, as set forth in CS24–30 and CS25–30, their standard of practice, effective from July 1, 1930. First revision and consolidation.—On March 25, 1942, the Inter-

First revision and consolidation.—On March 25, 1942, the Interdepartmental Screw Thread Committee,⁵ recognizing that the Commercial Standards CS24–30 and CS25–30 had been rendered obsolete by revisions since their publication, requested the development of revised and additional standards in line with the generally accepted commercial practice recorded in National Bureau of Standards

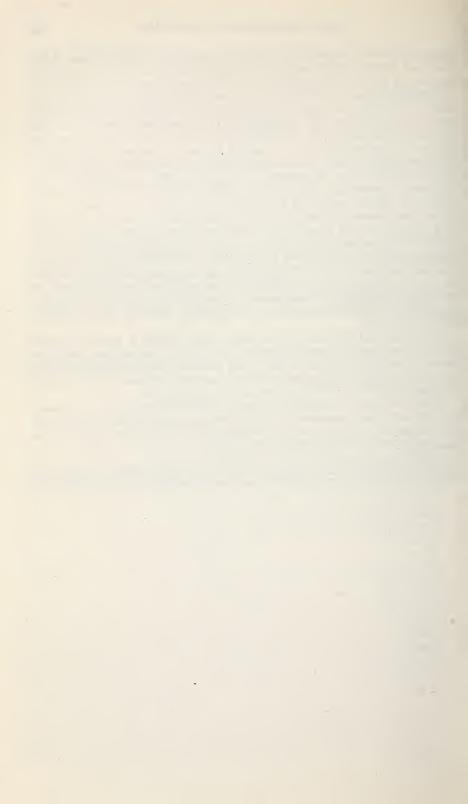
Handbook H28.

The National Bureau of Standards established a standing committee representing manufacturers, distributors, consumers, and laboratories, which reviewed, revised, and approved for circulation within the industry the Recommended Commercial Standard for Screw Threads and Tap Drill Sizes prepared by the Bureau.

Upon written acceptance by a predominant majority of users, distributors, and producers, as listed herein, announcement was made on November 10, 1942, that the standard would become effective for

new production from February 10, 1943.

⁵ The National Screw Thread Commission was abolished by Executive Order dated June 10, 1933. The Interdepartmental Screw Thread Committee was established September 14, 1939, by the Departments of War, Navy, and Commerce to promote uniformity in screw-thread standards in the Departments concerned.



ACCEPTANCE OF COMMERCIAL STANDARD

If acceptance has not previously been filed, this sheet properly filled in, signed and returned will provide for the recording of your organization as an acceptor of this commercial standard.

Division of Trade National Bureau o Washington, D. C.	Standards, f Standards,		·
Gentlemen:			
Having consider we accept the Co practice in the	ed the statements on the mmercial Standard CS	ne reverse side 824–43 as our	of this sheet, standard of
Production ¹	Distribution 1	Use 1	Testing ¹
of screw threads ar	d tap-drill sizes.		
	a securing its general restanding committee essary.		
Signature of indivi	dual officer	(in ink)	·
	(Kindly typewrite or print the foll	lowing lines)	
Name and title of a	bove officer		
Organization	(Fill in exactly as it should be		
Street address			
City and State			
1 Please designate which g	roup you represent by drawing lines	s through the other th	ree. Please file sep-

¹ Please designate which group you represent by drawing lines through the other three. Please file separate acceptances for all subsidiary companies and affiliates which should be listed separately as acceptors. In the case of related interests, trade papers, colleges, etc., desiring to record their general approval, the words "in principle" should be added after the signature.

TO THE ACCEPTOR

The following statements answer the usual questions arising in con-

nection with the acceptance and its significance:

1. Enforcement.—Commercial standards are commodity specifications voluntarily established by mutual consent of those concerned. They present a common basis of understanding between the producer, distributor, and consumer and should not be confused with any plan of governmental regulation or control. The United States Department of Commerce has no regulatory power in the enforcement of their provisions, but since they represent the will of the interested groups as a whole, their provisions through usage soon become established as trade customs, and are made effective through incorporation into sales contracts by means of labels, invoices and the like.

2. The acceptor's responsibility.—The purpose of commercial standards is to establish for specific commodities, nationally recognized grades or consumer criteria and the benefits therefrom will be measureable in direct proportion to their general recognition and actual use. Instances will occur when it may be necessary to deviate from the standard and the signing of an acceptance does not preclude such departures; however, such signature indicates an intention to follow the commercial standard where practicable, in the production, distribu-

tion, or consumption of the article in question.

3. The Department's responsibility.—The major function performed by the Department of Commerce in the voluntary establishment of commercial standards on a Nation-wide basis is fourfold: first, to act as an unbiased coordinator to bring all interested parties together for the mutually satisfactory adjustment of trade standards; second, to supply such assistance and advice as past experience with similar programs may suggest; third, to canvass and record the extent of acceptance and adherence to the standard on the part of producers, distributors, and users; and fourth, after acceptance, to publish and promulgate the standard for the information and guidance of buyers and sellers of the commodity.

4. Announcement and promulgation.—When the standard has been endorsed by a satisfactory majority of production or consumption in the absence of active, valid opposition, the success of the project is announced. If, however, in the opinion of the standing committee or the Department of Commerce, the support of any standard is inadequate, the right is reserved to withhold promulgation and publication.

ACCEPTORS

The organizations and individuals listed below have accepted these dimensions as their standard of practice in the production, distribution, and use of screw threads and tap-drill sizes. Such endorsement does not signify that they may not find it necessary to deviate from the standard, nor that producers so listed guarantee all of their products in this field to conform with the requirements of this standard. Therefore specific evidence of conformity should be obtained where required.

ASSOCIATIONS

Allied Building Metal Industries, New York, N. Y.

Association of Engineers, American

Chicago, Ill.

American Institute of Bolt, Nut, & Cleveland, Rivet Manufacturers, Ohio.

American Railway Engineering Association, Chicago, Ill. (In Principle.)
American Supply & Machinery Manu-

facturers' Association, Inc., burgh, Pa. (In Principle.) ssociated General Contracto

Associated Contractors America, Inc., Washington, D. C. Manufacturers Standardization Society

of the Valve & Fittings Industry, New York, N. Y.

National Association Master Plumbers, New York, N. Y. National Retail Hardware Association,

Indianapolis, Ind. National Screw Machine Products As-

sociation, Cleveland, Ohio. Southern Hardware Jobbers Associ-

ation, Atlanta, Ga.

Southern Supply & Machinery Distributors' Association, Inc., Atlanta,

FIRMS

Accurate Tool Co., Detroit, Mich. Acme Machine Tool Co., The, Cincinnati, Ohio.

Adams Co., The, Dubuque, Iowa. Aero Supply Manufacturing Co., Inc.,

Corry, Pa. Aircooled Motors Corporation, Syracuse, N. Y.

Ajax Bolt & Screw Co., Detroit, Mich. Allen Manufacturing Co., The, Hartford, Conn.

Almond Manufacturing Co., T. Ashburnham, Mass.

Aluminum & Brass Co., Lockport:

Aluminum Company of America, Pittsburgh, Pa.

American Bridge Co., Pittsburgh, Pa. Locomotive Co., Schenec-American tady, N. Y.

Manganese Bronze American Holmesburg, Philadelphia, Pa. American Seating Co., Grand Rapids,

Mich. American Screw Co., Providence, R. I.

Armstrong Manufacturing Co., The, Bridgeport, Conn.

Products Automatic ration, New York, N. Y.
Atlantic Machine Screw Co., S. Boston,

Atlas Bolt & Screw Co., The, Cleve-

land, Ohio. Atlas Copper & Brass Manufacturing Co., Chicago, Ill.

Autocar Co., Ardmore, Pa. Automatic Machinery Manufacturing Corporation, Bridgeport, Conn. Automatic Products Co., Milwaukee,

Autoscrew Co., New York, N. Y. Avey Drilling Machine Co., The, Covington. Ky.

Babson-Dow Manufacturing Co., Roxbury, (Boston) Mass.

Baldwin Locomotive Works, The, Philadelphia, Pa.

Bard Manufacturing Co., Royersford,

Bath & Co., John, Worcester, Mass. Bausch & Lomb Optical Co., Rochester, N. Y.

Bausch Machine Tool Co., Springfield, Mass.

Bayonne Bolt Corporation, Bayonne, Beard Tool Co., L. O., Lancaster, Pa.

Bell Co., Inc., The David, Buffalo, Curtis Screw Co., Inc., Buffalo, N. Y.

Dallett Co., The, Philadelphia, Pa.

Bethlehem Steel Co., Lebanon, Pa. Bicknell Manufacturing Co., Rockland, Maine

Biglow & Co., Inc., L. C., New York, N. Y.

Billings & Spencer Co., The, Hartford,

Bommer Spring Hinge Co., Brooklyn, N. Y.

Machine Works Co., Lynn, Boston Mass.

Botwinik Brothers, Inc., Hamden, New Haven, Conn.

Brightman Nut & Manufacturing Co., Sandusky, Ohio.

Brill Co., The J. G., Philadelphia, Pa. Brown Bag Filling Machine Co., The, Fitchburg, Mass. Brown & Sharpe Manufacturing Co.,

Providence, R. I. Brown-Wales Co., Boston, Mass. Buckeye Traction Ditcher Co., The, Findlay, Ohio. Buda Co., The, Harvey, Ill.

Buerk Tool Works, Buffalo, N. Y. Buffalo Bolt Co., North Tonawanda,

N. Y. Camden Forge Co., Camden, N. J. Cap Screw & Nut Co. of America, Inc.,

New York, N. Y. Carey Machinery & Supply Co., Balti-

more, Md. Central Screw Co., Chicago, Ill. Chain Belt Co., Milwaukee, Wis. Chatillon & Sons, John, New York,

N. Y.

Chicago, Rock Island & Pacific Railway Co., Chicago, Ill. Chicago Screw Co., The, Chicago, Ill. Chrysler Corporation, Detroit, Mich. Cincinnati Planer Co., The, Cincinnati,

Ohio.

City Engineering Co., The, Dayton, Ohio.

Clark, Jas., Jr., Paterson, N. J. Clark Bros. Bolt Co., Milldale, Conn. Clark Metal Products, Inc., Bridgeport, Conn. Clendenin Bros. Inc., Baltimore, Md.

Cleveland Automatic Machine The, Cleveland, Ohio. Cleveland Cap Screw Co., The, Cleve-

land, Ohio.

Cleveland Die & Manufacturing Co., The, Cleveland, Ohio.

Columbus Bolt Works Co., The, Columbus, Ohio.

Commonwealth Brass Corporation, Detroit, Mich.

Comtor Co., The, Waltham, Mass. Connecticut Tool & Engineering Co., Bridgeport, Conn. Continental Screw Co., New Bedford,

Cox & Sons Co., The, Bridgeton, N. J. Crane Co., Chicago, Ill.

Dardelet Threadlock Corporation, Detroit, Mich.

Davis & Hemphill, Elkridge, Md. Defiance Machine Works, Inc., Defiance, Ohio.

Detroit Nut Co., Inc., Detroit, Mich. Plating Industries, Detroit, Detroit Mich.

Detroit Tap & Tool Co., Detroit, Mich. Doehler Die Casting Co., Batavia, N. Y. Dravo Corporation Engineering Works Division, Pittsburgh, Pa.

Eastern Machine Screw Corporation, The, New Haven, Conn.

Eastman Kodak Co., Hawk-Eye Division, Rochester, N. Y.
Eastwood-Nealley Corp., Belleville, N. J.
Economy Engineering Co., The, Willoughby, Ohio.
Ekstrom, Carlson & Co., Rockford, Ill.

Electric Boat Co., Groton, Conn. Elterich Co., Chas., New York, N. Y.

(In Principle.)

Industries, Emery Inc., Cincinnati, Ohio.

Engineers Club of Philadelphia, Philadelphia, Pa. (In Principle.) Erie Bolt & Nut Co., Erie, Pa. Essley Machinery Co., The

Chicago, Ill. (In Principle.) Fairbanks, Morse & Co., Beloit, Wis.

Federal Products Corporation, Providence, R. I.
Federal Screw Works, Detroit, Mich. Ferry Cap & Set Screw Co., The, Cleve-

land, Ohio. Firestone Steel Products Co., Akron, Ohio.

Firestone Tire & Rubber Co., Akron, Ohio.

Flannery Bolt Co., Bridgeville, Pa. Fox Munitions Corporation, Philadelphia, Pa.

Foxboro Co., The, Foxboro, Mass. General Engineering Works, Chicago,

General Electric Co., Schenectady, N. Y. Manufacturing Co., The, General Waterbury, Conn.

General Motors Corporation, Detroit, Mich.

Geometric Tool Co., The, New Haven, Conn.

Gibbs & Cox, Inc., New York, N. Y. Gisholt Machine Co., Madison, Wis.

Globe Products Co., The, Cleveland, Ohio.

Grabler Manufacturing Co., The, Cleveland, Ohio. Grant Manufacturing & Machine Co.,

The, Bridgeport, Conn. Graves Elevator Co., Inc., Rochester,

N.Y Greenfield Tap & Die Corporation, Greenfield, Mass.

Greenlee Bros. & Co., Rockford, Ill.

Chicago, Ill. Gurley, W. & L. E., Troy, N. Y. Haines Gauge Co., Inc., Philadelphia, Pa.

Hardware Products Co., Inc., Boston, Mass.

Harper Co., The H. M., Chicago, Ill. Hartford Machine Screw Co., Hartford, Conn.

Hassall, Inc., John, Brooklyn, N. Y. Haynes Stellite Co., Kokomo, Ind. Hodell Chain Co., The, Cleveland, Ohio.

Hood Co., R. H., Philadelphia, Pa. Hooper Co., Inc., F. X., Glenarm, Md. Hudson Motor Car Co., U. S. Naval

Ordnance Plant, Center Line, Mich. Illinois Iron & Bolt Co., Carpentersville,

Imsande Screw Products Co., Cincin-

nati, Ohio.

Indicating Calipers Corporation, New York, N. Y. International Business Machines Cor-poration, Endicott, N. Y.

International Harvester Co., Chicago,

International Machine Tool Corpora-tion, Foster Division, Elkhart, Ind. International-Stacey Corporation, International Derrick & Equipment Division, Columbus, Ohio. Isaacson Iron Works, Seattle, Wash.

Iverson & Laux, Inc., Chicago, Ill. Jacobs Aircraft Engine Co., Plant No. 1,

Pottstown, Pa.

Jeffrey Manufacturing Co., The, Columbus, Ohio. Johnson Automatics Manufacturing Co.,

Providence, R. I. Johnson Rule Manufacturing Co., E. P.,

Chicago, Ill. Johnston & Jennings Co., The, Cleveland, Ohio.

Jones & Lamson Machine Co., Springfield, Vt.

Jordan Machine Products, Inc., Detroit, Mich.

Judson-Pacific Co., San Francisco, Calif. Kaufman Manufacturing Co., L. J., Manitowoc, Wis.

King Engineering Corporation, Ann Arbor, Mich.
Kinner Motors, Inc., Glendale, Calif.
Kramer Co., C. P., Chicago, Ill.

Lamson & Sessions Co., The, Cleveland,

Ohio.

Landis Machine Co., Waynesboro, Pa. Lanman Co., The E. B., East Chicago, Ind.

Larson Tool & Stamping Co., Attleboro, Mass. Lima Locomotive Works, Inc., Lima,

Ohio. Link-Belt Ordnance Co., Chicago, Ill.

Lionel Corporation, The, Irvington, N. J. Los Angeles Testing Laboratory, Los Angeles, Calif.

Grimm Hardware Co., Inc., W. H., Lundberg Screw Products Co., Lansing, Mich.

Machined Products Co., Louisville, Ky. MacLean-Fogg Lock Nut Co., Chicago, 111.

Macy & Co., Inc., R. H., New York, N. Y. Maine Steel, Inc., South Portland,

Maine.

Mann & Co., Hutchinson, Kans.

Maryland Bolt & Nut Co.. Baltimore, Md.
Mattatuck Manufacturing Co.,

Waterbury, Conn.

Merrill Brothers, Maspeth, N. Y.

Mid-West Screw Products Co., Louis, Mo. Milled Screw Products Co., Chicago,

Ill. Milton Manufacturing Co., The, Milton,

Pa.

Mitchell Engineering Co., The, Spring field, Ohio.

Modern Tool Works, Rochester, N. Y. Moore, Inc., George W., Boston, Mass. Morrow Screw & Nut Co., Ltd., Ingersoll, Ontario, Canada. Morse Twist Drill & Machine Co., New

Bedford, Mass.

Mueller Co., Decatur, Ill. Murchey Machine & Tool Co., Detroit, Mich.

Napoleon Products Co., The, Napoleon, Ohio.

National Acme Co., The, Cleveland, Ohio.

National Brass Co., Grand Rapids, Mich.

National Lock Co., Rockford, Ill. National Machine Products Co., De-

troit, Mich. National Screw & Manufacturing Co.,

The, Cleveland, Ohio. New Britain Machine Co., The New

Britain, Conn. New York Air Brake Co., The, Watertown, N. Y.

New York Central System, New York,

N. Y. Nilson Machine Co., The A. H.,

Bridgeport, Conn. North & Judd Manufacturing Co.,

New Britain, Conn. Northwest Automatic Products Corpo-

ration, Minneapolis, Minn. Northwest Bolt & Nut Co., Seattle, Wash.

Ohio Brass Co., The, Mansfield, Ohio. Oliver Iron & Steel Corporation, Pitts-

burgh, Pa. Osgood Engineering Co., Boston, Mass. Ottemiller Co., The Wm. H., York, Pa. Pacific Car & Foundry Co., Renton,

Wash. Packard Motor Car Co., Detroit, Mich.

Palnut Co., The, Irvington, N. J. Parker Wire Goods Co., Worcester, Mass.

Pawtucket Manufacturing Co., Pawtucket, R. I.

Peck, Stow & Wilcox Co., Southington, Conn.

Peerless Manufacturing Corporation,

Louisville, Ky. Penn Screw & Machine Works, Philadelphia, Pa.

Perry Fay Co., Elyria, Ohio.

Pheoll Manufacturing Co., Chicago, Ill. Philadelphia Hardware & Malleable Iron Works, Inc., Philadelphia, Pa.

Pioneer Engineering & Manufacturing

Co., Detroit, Mich.

Pioneer Pump & Manufacturing Co., Detroit, Mich.

Pittsburgh Screw & Bolt Corporation,

Pittsburgh, Pa.
Potter Tool & Machine Works, Inc.,
New York, N. Y.

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